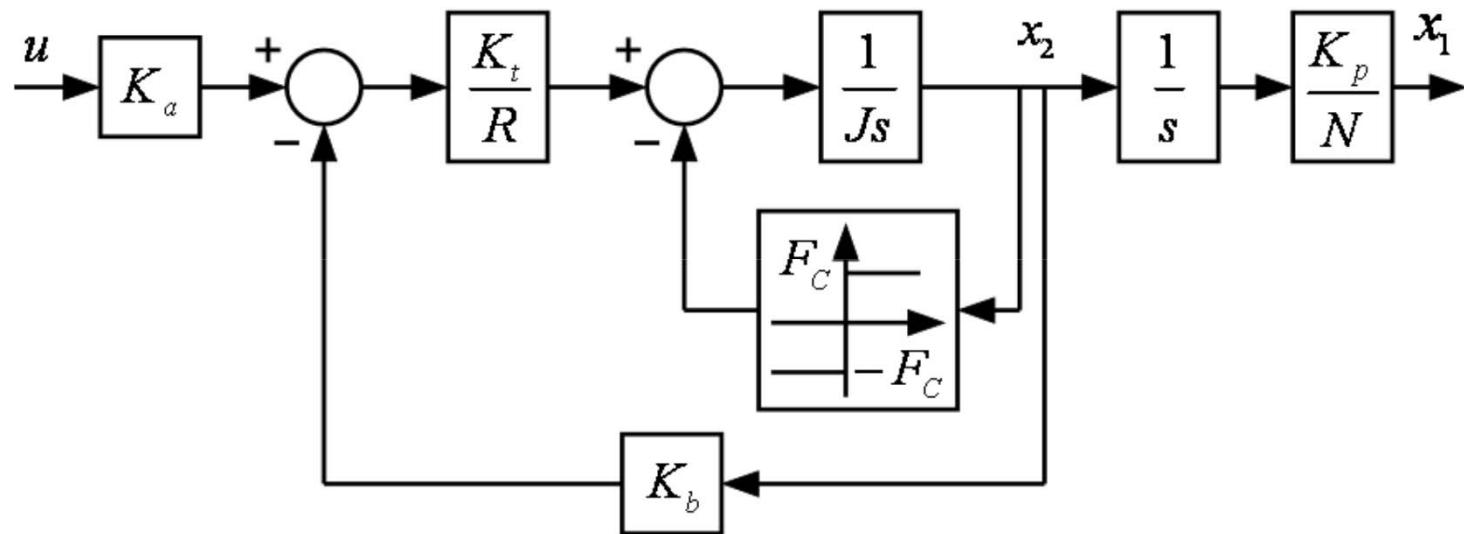


Topics on Labs

- Limit Cycle in Motor Control System
- Magnetic Levitation
- Ball & Beam

Limit Cycle in Motor Control System

Motor



$$\dot{x}_1 = \frac{K_p}{N} x_2$$

$$\dot{x}_2 = \frac{K_t}{R} \frac{1}{J} (K_a u - K_b x_2) = -\frac{K_t K_b}{R J} x_2 + \frac{K_t K_a}{R J} u$$

Motor

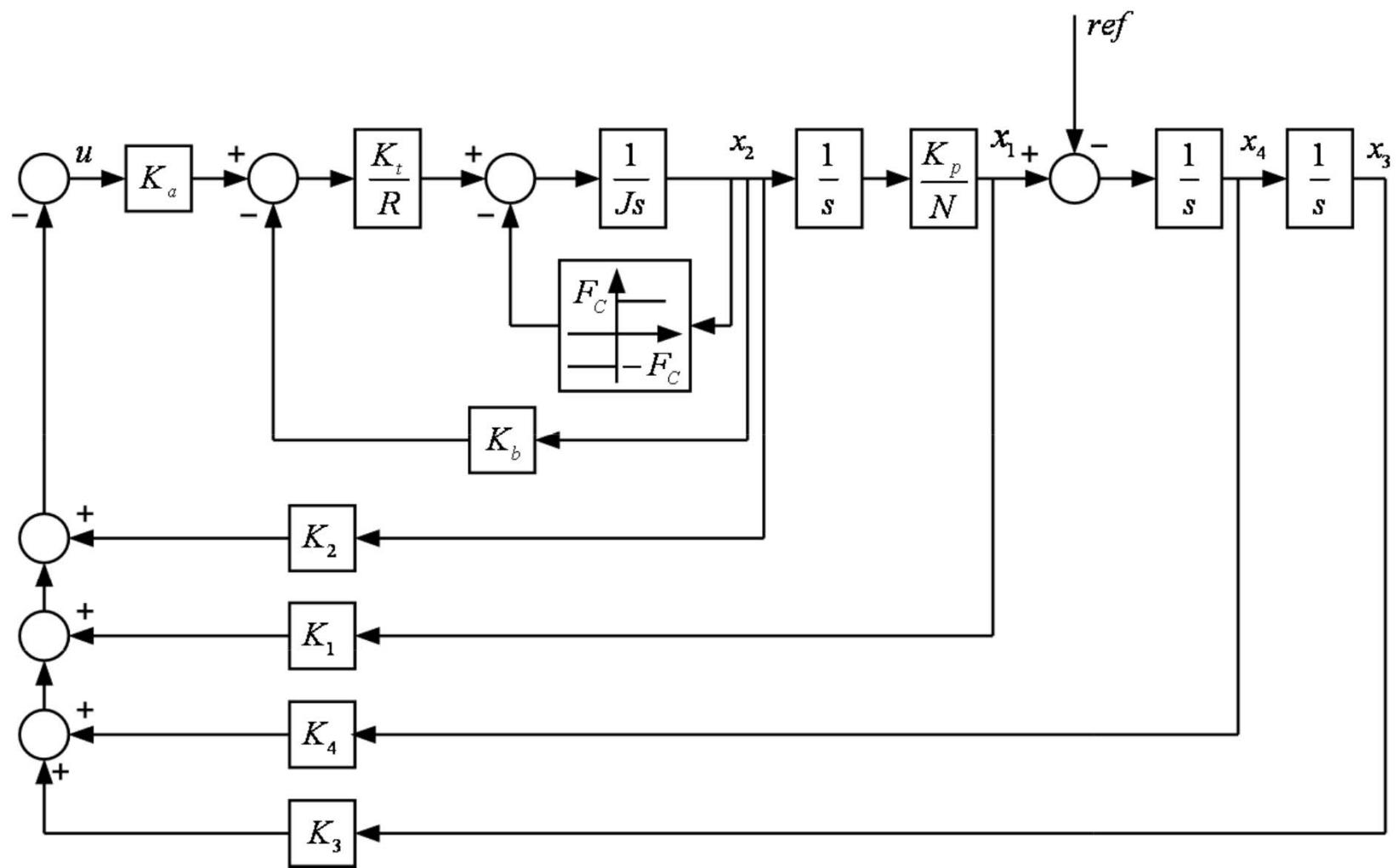
$$\dot{x} = Ax + Bu$$

$$y = Cx$$

$$A = \begin{bmatrix} 0 & \frac{K_p}{N} \\ 0 & -\frac{K_t K_b}{RJ} \end{bmatrix}, B = \begin{bmatrix} 0 \\ \frac{K_t K_a}{RJ} \end{bmatrix}$$

$$C = [1 \ 0]$$

Tracking Controller



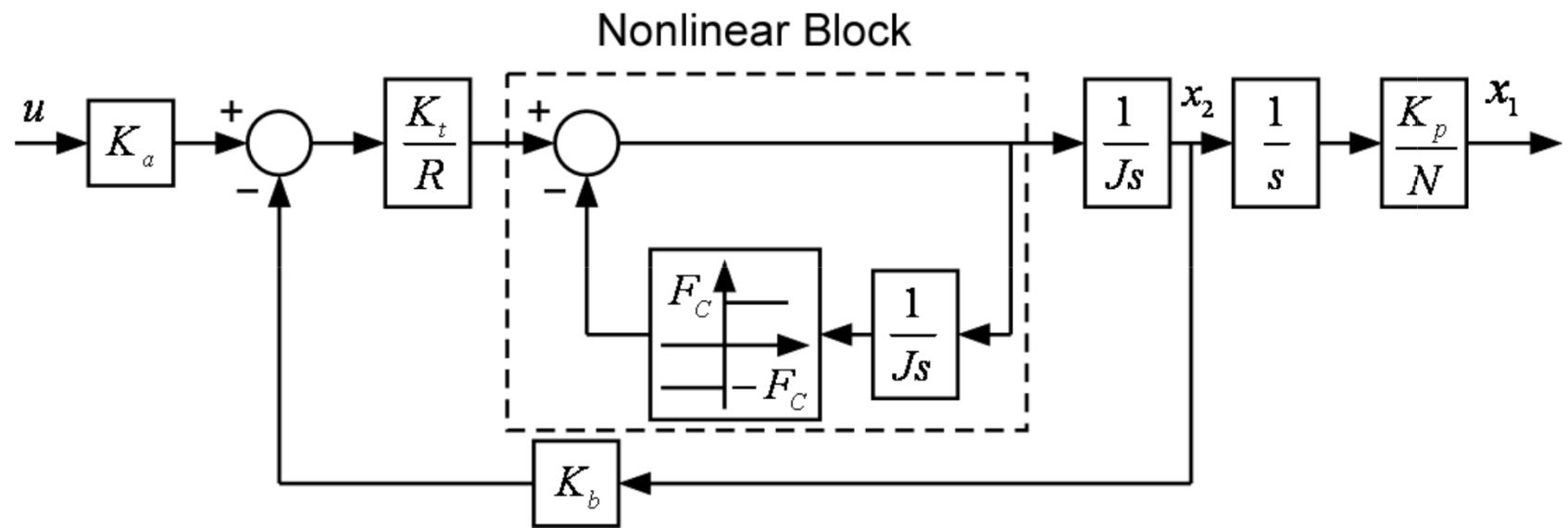
Tracking Controller

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} A & 0 & 0 \\ & 0 & 0 \\ 0 & 0 & 1 \\ 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} B \\ 0 \\ 0 \\ 0 \end{bmatrix} u + \begin{bmatrix} 0 \\ 0 \\ 0 \\ -1 \end{bmatrix} ref$$

$$u = -(K_1 x_1 + K_2 x_2 + K_3 x_3 + K_4 x_4)$$

$$J = \int_0^{\infty} \left([x_1 \quad x_2 \quad x_3 \quad x_4] Q \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + R u^2 \right) dt$$

Motor



$$F_C = (0.5A) \times K_t = 0.0333$$

Describing Function

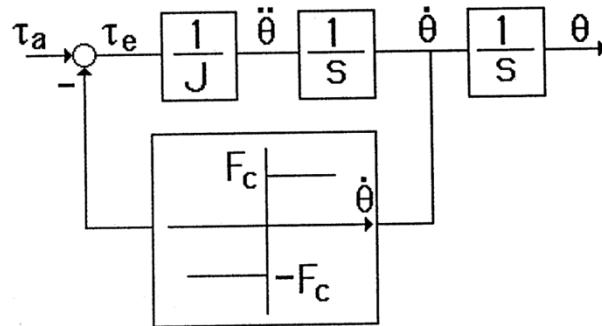


Figure B.1. Rotating member with Coulomb friction

Because of Coulomb friction the effective accelerating or decelerating torque τ_e is related to the applied torque τ_a through the equation

$$\tau_e = \tau_a \pm F_c \quad (B.1)$$

where F_c is defined as in Figure B.1. From Newton's law of motion,

$$\tau_a = F_c + J\ddot{\theta} \quad \text{for } \dot{\theta} > 0 \quad (B.2)$$

$$\tau_a = -F_c + J\ddot{\theta} \quad \text{for } \dot{\theta} < 0 \quad (B.3)$$

$$\tau_a = M \sin \omega t . \quad (B.5)$$

The corresponding steady-state wave forms are sketched in Figure B.2 and B.3. The discontinuities of the τ_e wave correspond to zeros of the $\dot{\theta}$ wave because the frictional torque F_c changes sign at those instants. $\ddot{\theta}(t)$ passes through zero at

$$\omega t = n \pi - \alpha , \quad n = 0, 1, 2, \dots , \quad (B.6)$$

while

$$\tau_a = F_c \quad \text{at} \quad \omega t = \alpha .$$

It follows that

$$\alpha = \sin^{-1} \lambda \quad (B.7)$$

where

$$\lambda = F_c / M .$$

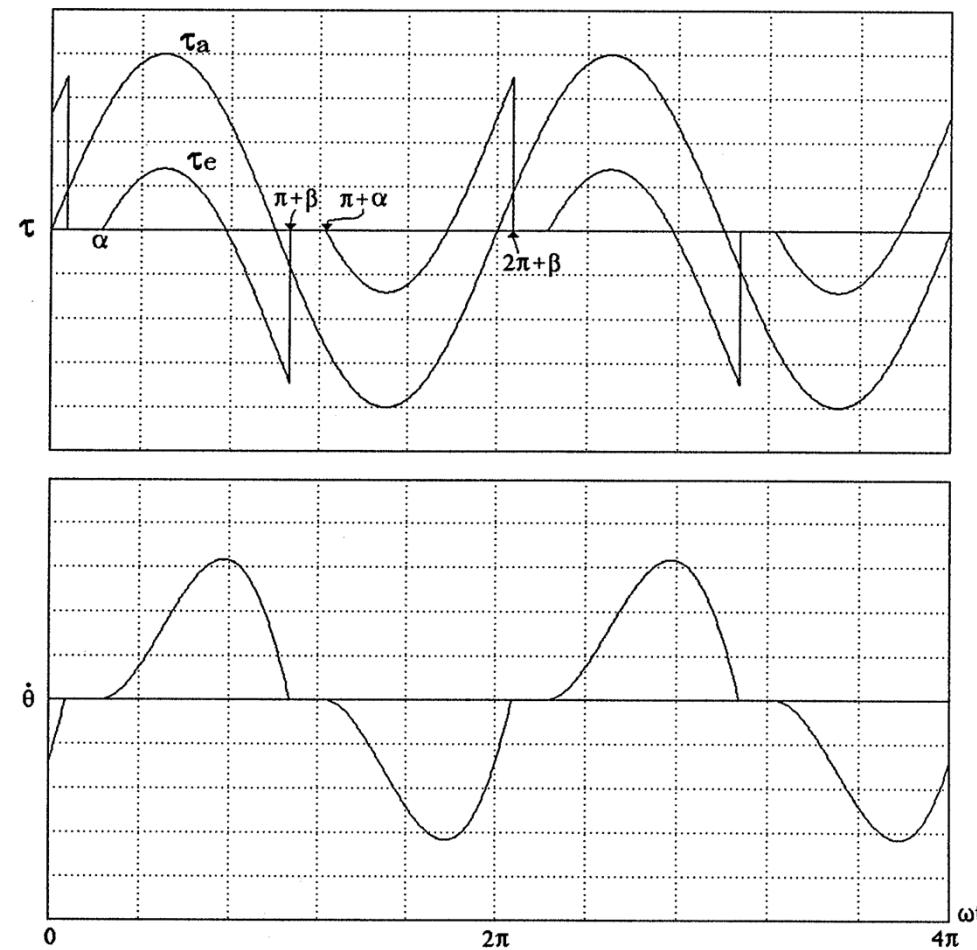


Figure B.2. Steady state wave forms with dead zones

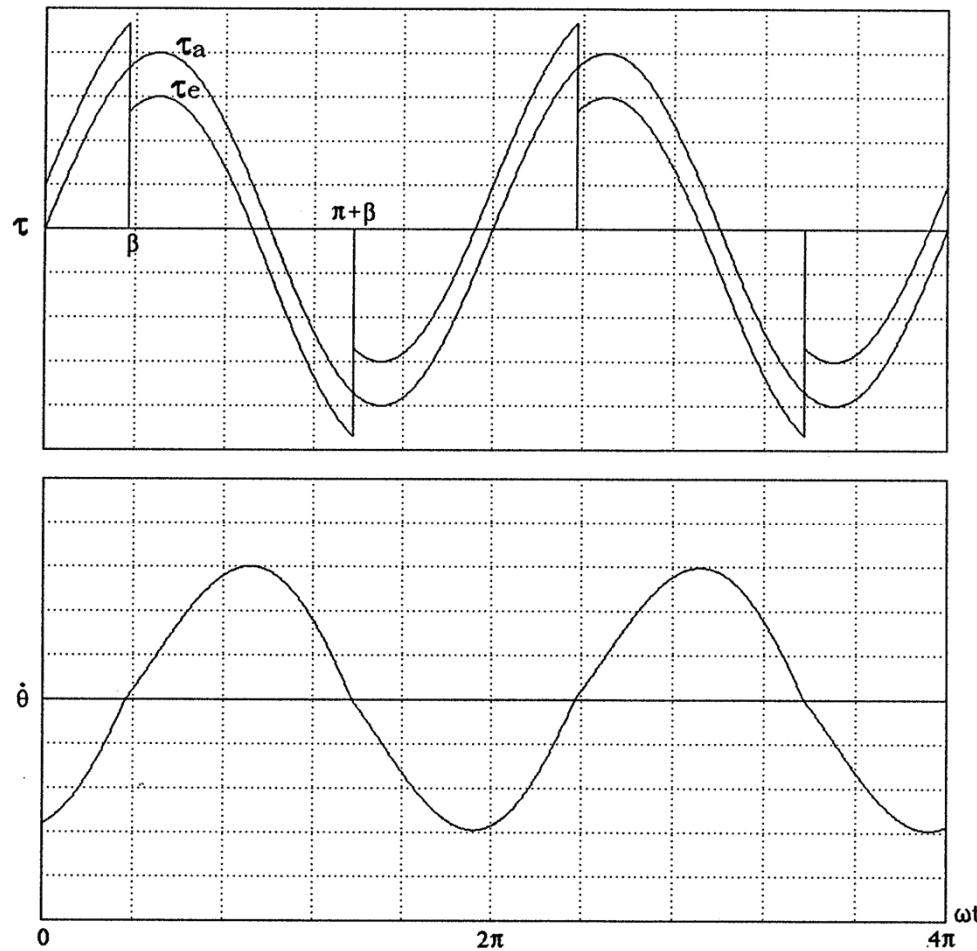


Figure B.3. Steady state wave forms without dead zones

Case 1: With dead zones. $\alpha > \beta$

As can be seen in Figure B.2, the following steady-state condition exists.

$$\int_{\alpha}^{\pi+\beta} (M \sin \omega t - F_c) d\omega t = 0 \quad (\text{B.8})$$

or

$$\int_{\alpha}^{\pi+\beta} (\sin \theta - \lambda) d\theta = 0$$

Simplifying eq. (B.8),

$$\beta = \frac{\cos \beta}{\lambda} + \frac{\sqrt{1-\lambda^2}}{\lambda} - \pi + \alpha \quad (\text{B.9})$$

Case 2: No dead zone. $\alpha \leq \beta$

From Figure B.3,

$$\int_{\beta}^{\pi+\beta} (M \sin \omega t - F_c) d\omega t = 0 \quad (\text{B.10})$$

or

$$\int_{\beta}^{\pi+\beta} (\sin \theta - \lambda) d\theta = 0$$

When eq. (B.10) is simplified,

$$\beta = \cos^{-1}(\pi \lambda / 2) \quad (\text{B.11})$$

Case 1: With dead zones, $\lambda > \lambda_c$ or $\alpha > \beta$

$$a_1 = \frac{1}{\pi} \int_{\alpha}^{\pi+\beta} M(\sin \omega t - \lambda) \sin(\omega t) d(\omega t) + \frac{1}{\pi} \int_{\pi+\alpha}^{2\pi+\beta} M(\sin \omega t + \lambda) \sin(\omega t) d(\omega t) \quad (\text{B.16})$$

$$= (M/\pi)[\pi - (\alpha - \beta) - \sin \alpha (\cos \alpha + \cos \beta) - \cos \beta (\sin \alpha + \sin \beta)]$$

Similarly

$$b_1 = (M/\pi)(\sin \alpha + \sin \beta)^2 \quad (\text{B.17})$$

Case 2: No dead zone, $\lambda \leq \lambda_c$ or $\alpha \leq \beta$

$$a_1 = \frac{2}{\pi} \int_{-\alpha}^{\beta} M(\sin \omega t + \lambda) \sin(\omega t) d(\omega t) + \frac{2}{\pi} \int_{\beta}^{\pi - \alpha} M(\sin \omega t - \lambda) \sin(\omega t) d(\omega t) \quad (\text{B.18})$$

$$= M(1 - 2\lambda^2)$$

Similarly

$$b_1 = 2M\lambda \sqrt{[(2/\pi)^2 - \lambda^2]} \quad (\text{B.19})$$

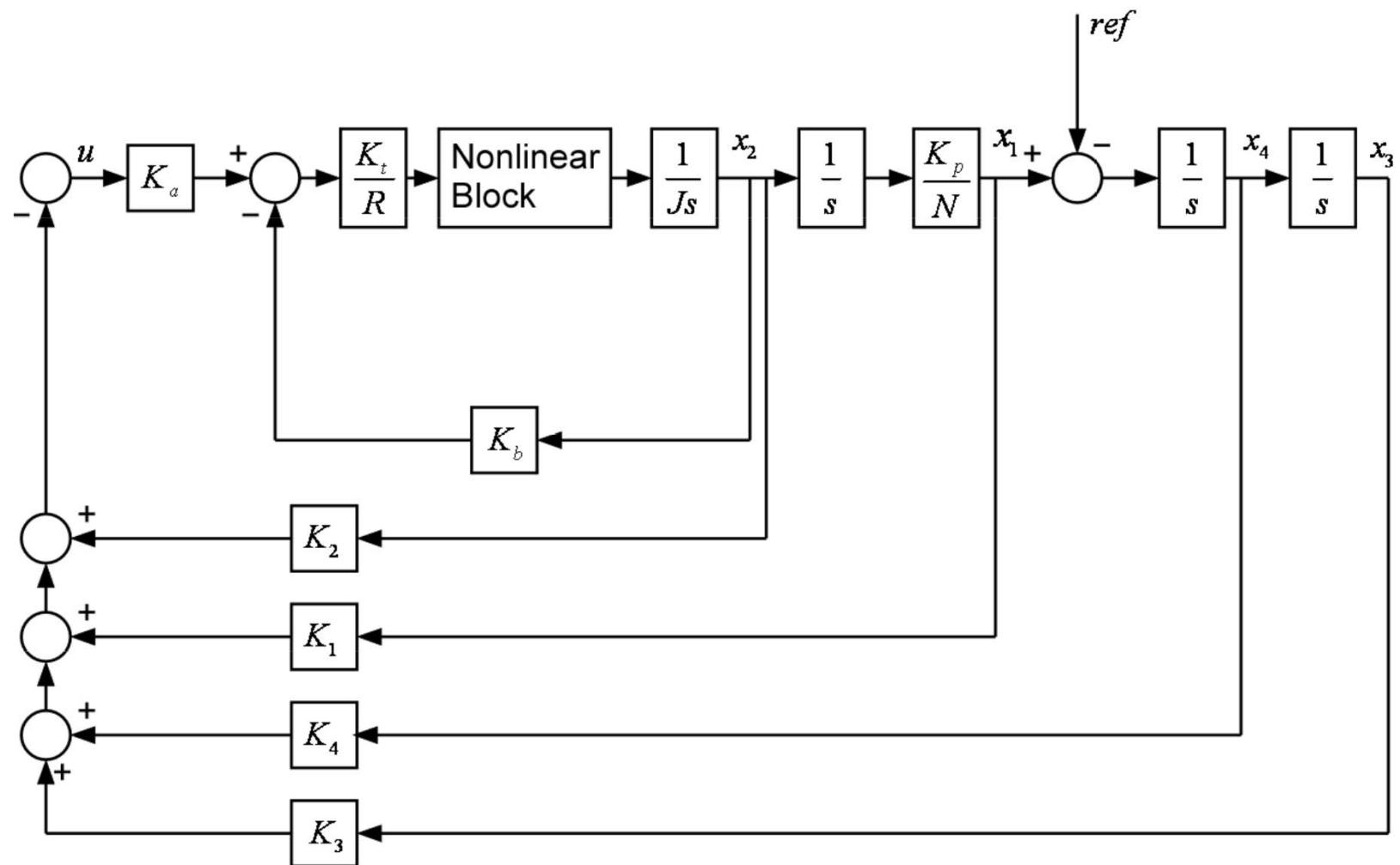
Then, the Describing Function, $N_{df}(M)$, is given by

$$N_{df}(M) = \frac{1}{M} \sqrt{a_1^2 + b_1^2} \exp(j\phi) \quad (\text{B.20})$$

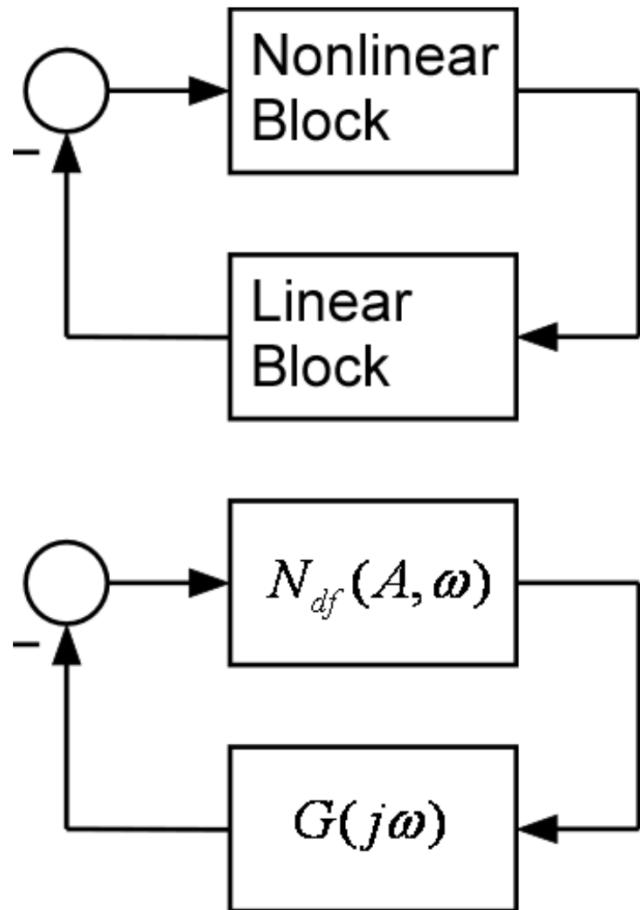
where

$$\phi = \tan^{-1}\left(\frac{b_1}{a_1}\right)$$

Tracking Controller



Limit Cycle Condition



$$1 + G(j\omega)N_{df}(A, \omega) = 0$$

$$G(j\omega) = -\frac{1}{N_{df}(A, \omega)}$$

Controller Design

$$A = [0 \ K_p/N; 0 \ -K_t * K_b / (R * J_m)]$$

$$B = [0; K_a * K_t / (J_m * R)]$$

$$C = [1 \ 0]$$

$$D = [0]$$

$$A_6 = [A \ [0 \ 0; 0 \ 0]; 0 \ 0 \ 0 \ 1; 1 \ 0 \ 0 \ 0];$$

$$B_6 = [B; 0; 0];$$

$$C_6 = [1 \ 0 \ 0 \ 0 \ 0 \ 0; 0 \ 0 \ 0 \ 0 \ 0 \ 1];$$

$$D_6 = [0; 0; 0];$$

$$Q_6 = [0 \ 0 \ 0 \ 0; 0 \ 0 \ 0 \ 0; 0 \ 0 \ 1000 \ 0; 0 \ 0 \ 0 \ 0];$$

$$R_6 = [1];$$

$$K_6 = lqr(A_6, B_6, Q_6, R_6)$$

Linear Block Nyquist Plot

$A_1 = [0 \ K_p/N \ ; 0 \ 0]$

$B_1 = [0; 1/(Jm)]$

$C_1 = [0 \ K_t * K_b / R \ 0 \ 0]$

$A_3 = [A_1 \ [0 \ 0; 0 \ 0]; 0 \ 0 \ 0 \ 1; 1 \ 0 \ 0 \ 0];$

$B_3 = [B_1; 0; 0]$

$C_3 = K_a * K_t * K_b / R + C_1$

$w = logspace(-1, 3, 500);$

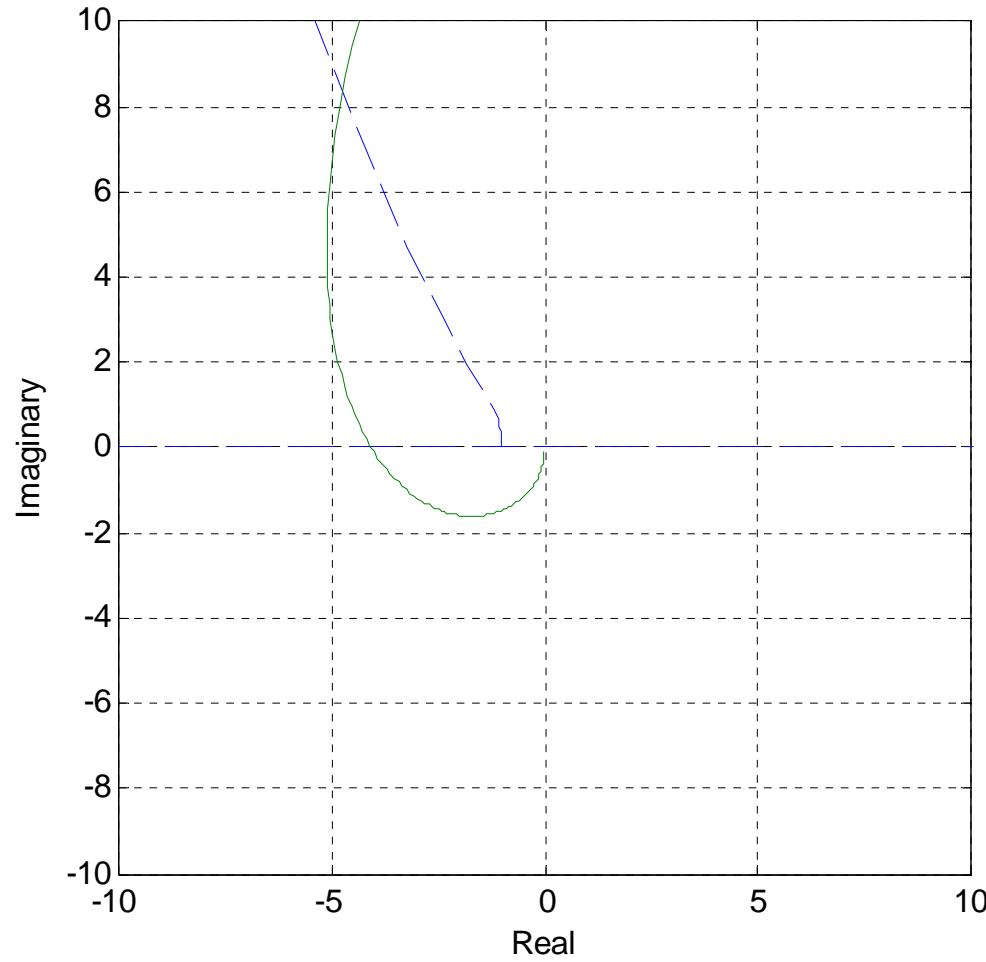
$[re, im] = nyquist(A_3, B_3, C_3, 0, 1, w);$

Linear Block Nyquist Plot

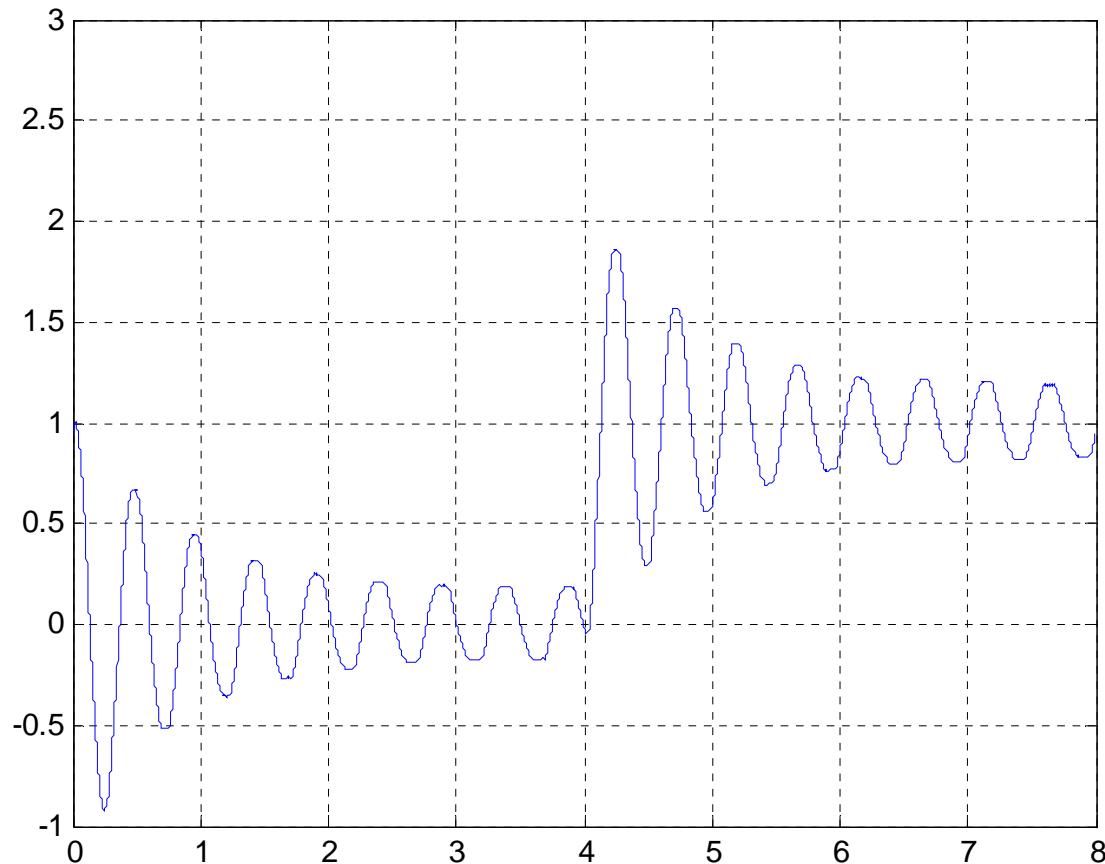
```
load -ascii desf
plot(desf(:,1),desf(:,2),'--',re,im);
axis([-10 10 -10 10])
axis('square')
xlabel('Real');
ylabel('Imaginary');
grid
```

K6

Linear Block Nyquist Plot



Limit Cycle in Response

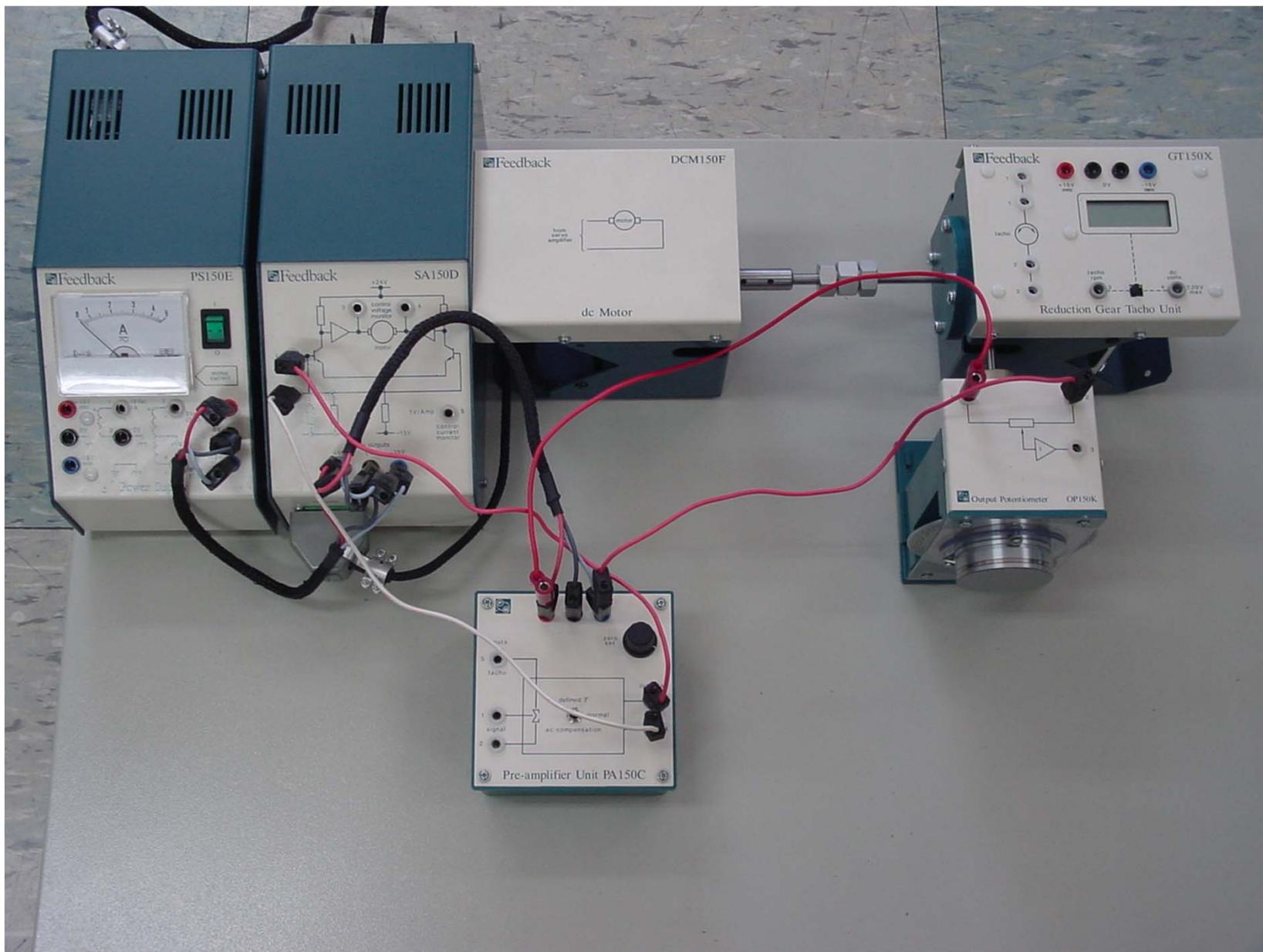


Exercise 1

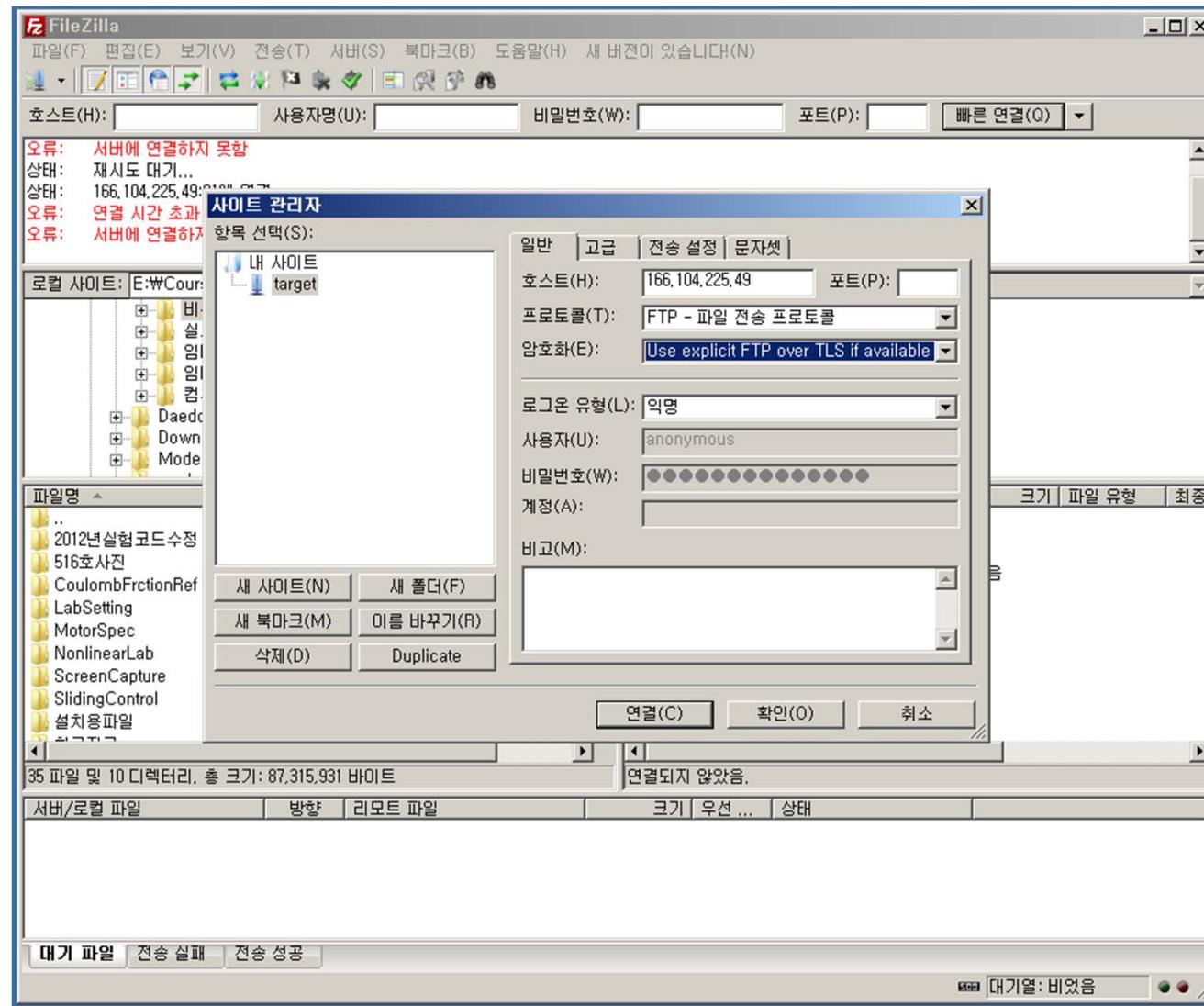
- 실험을 실행하여 리미트 사이클을 관찰하시오.
- Nyquist plot과 Describing function plot의 교차점에서 리미트 사이클의 주파수와 크기를 예측해 보고, 실험 결과와 비교해 보시오.
- MATLAB을 이용한 LQR 설계에서 Q값을 다르게 정하여 위의 실험을 반복하시오.(x1과 x2의 weight를 줄 경우 x2의 gain이 커져서 잡음의 영향이 커짐. X3의 weight는 0이 아니어야 함.)

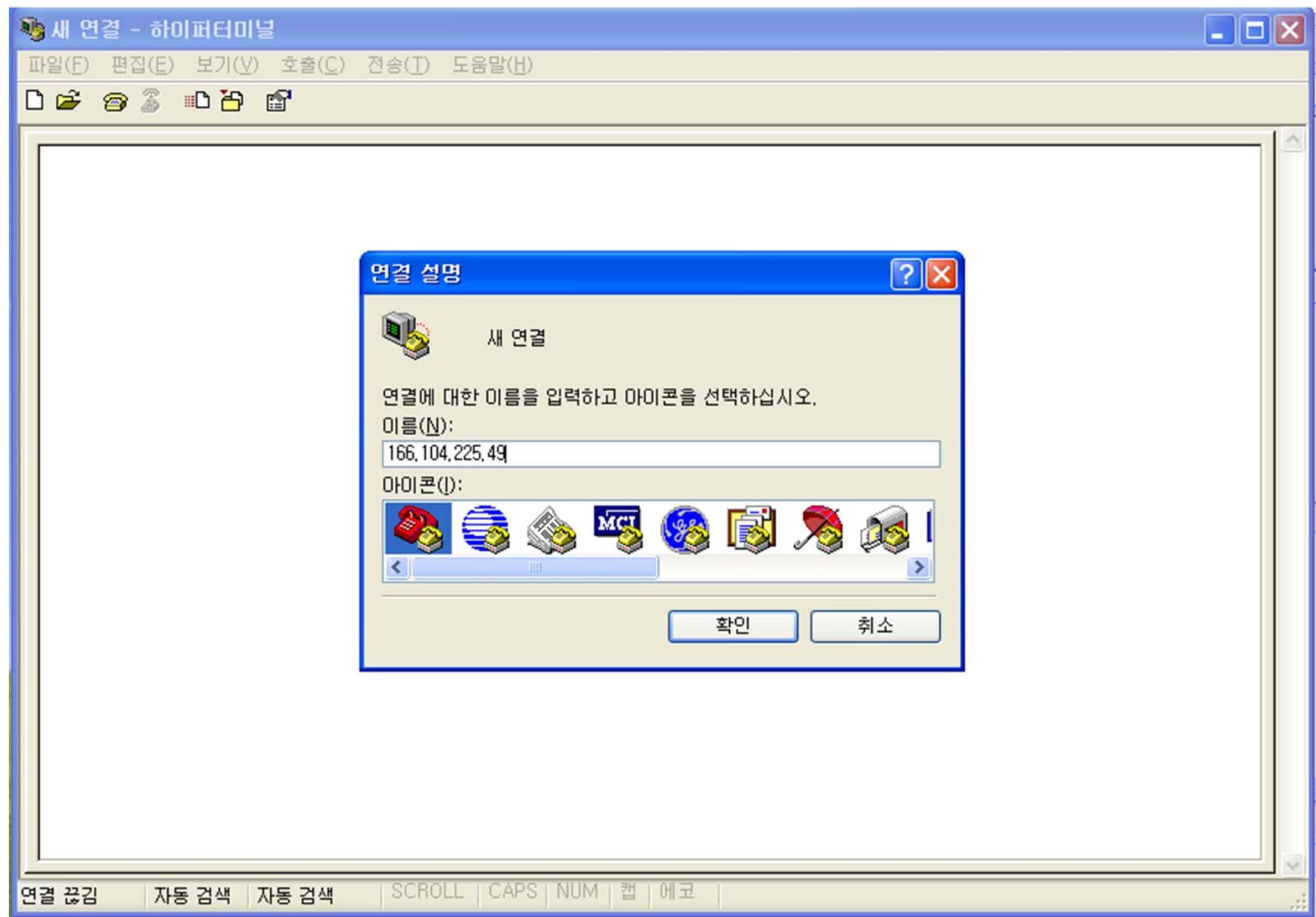
Exercise 1

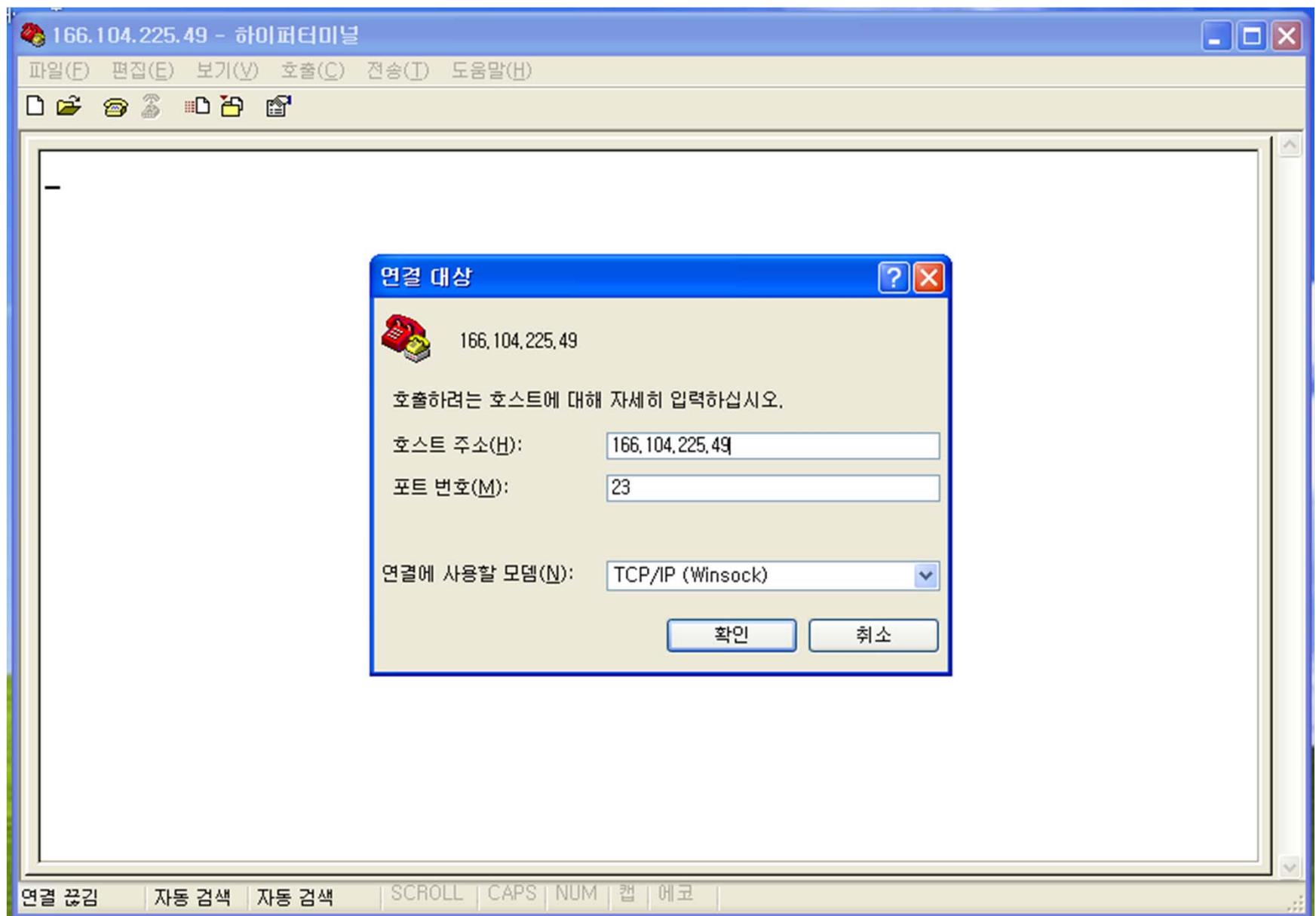
- Observe the limit cycle.
- Predict the frequency and amplitude of the limit cycle from the intersection of Nyquist plot and describing function curve. Compare the predicted results with the experiments.
- Repeat the above experiment with the different values of Q.(If you increase the weights of x_1 and x_2 , the influence of noise will be increased due to the large gain of x_2 . The weight of x_3 must be nonzero.)

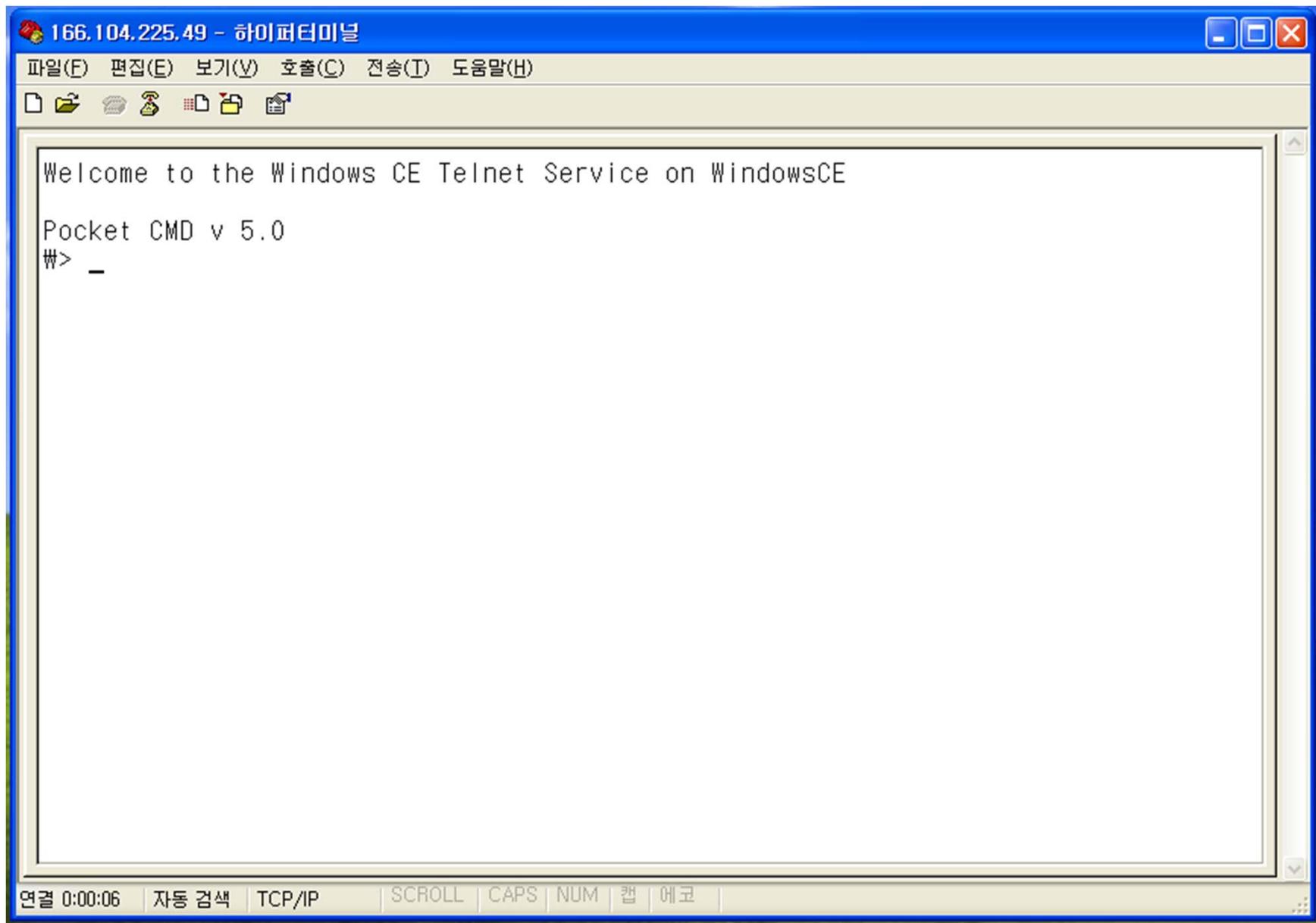


Connect FTP before experiment

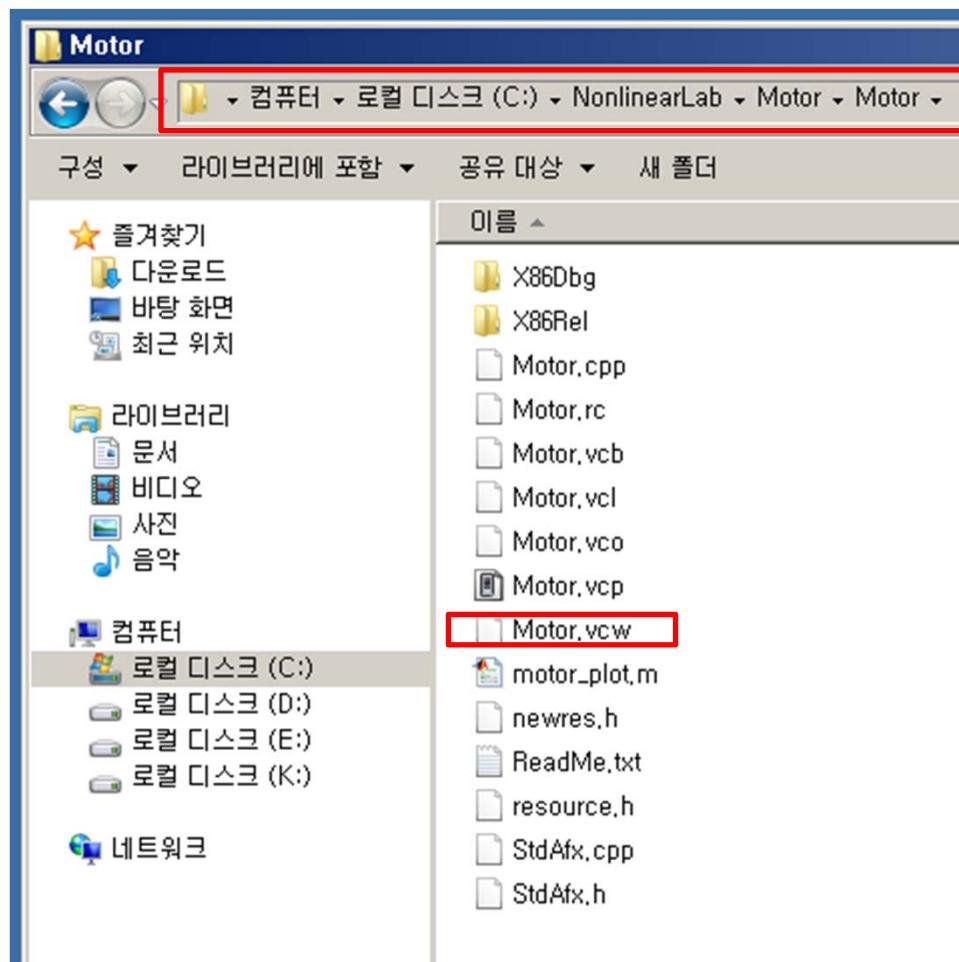




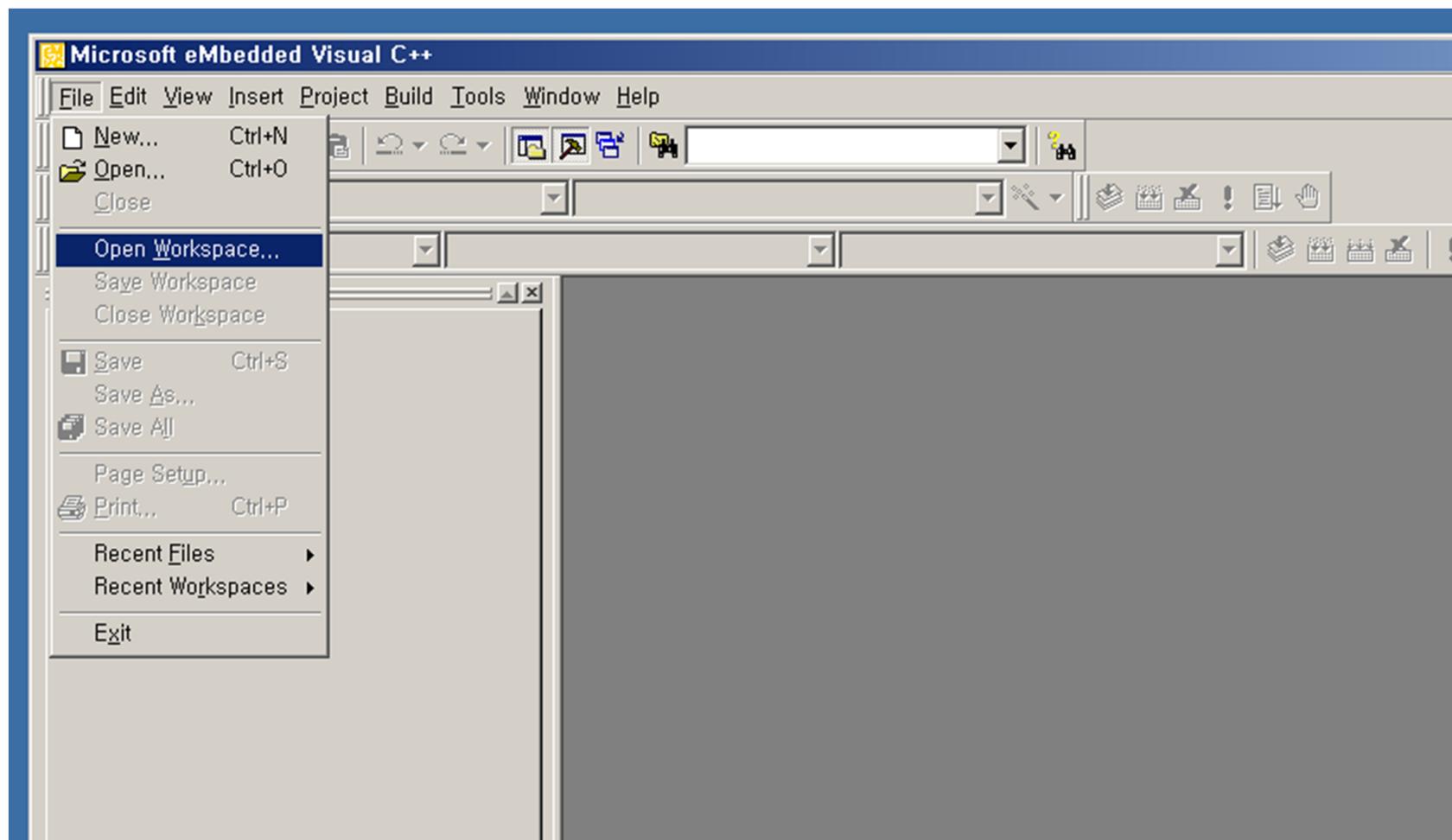




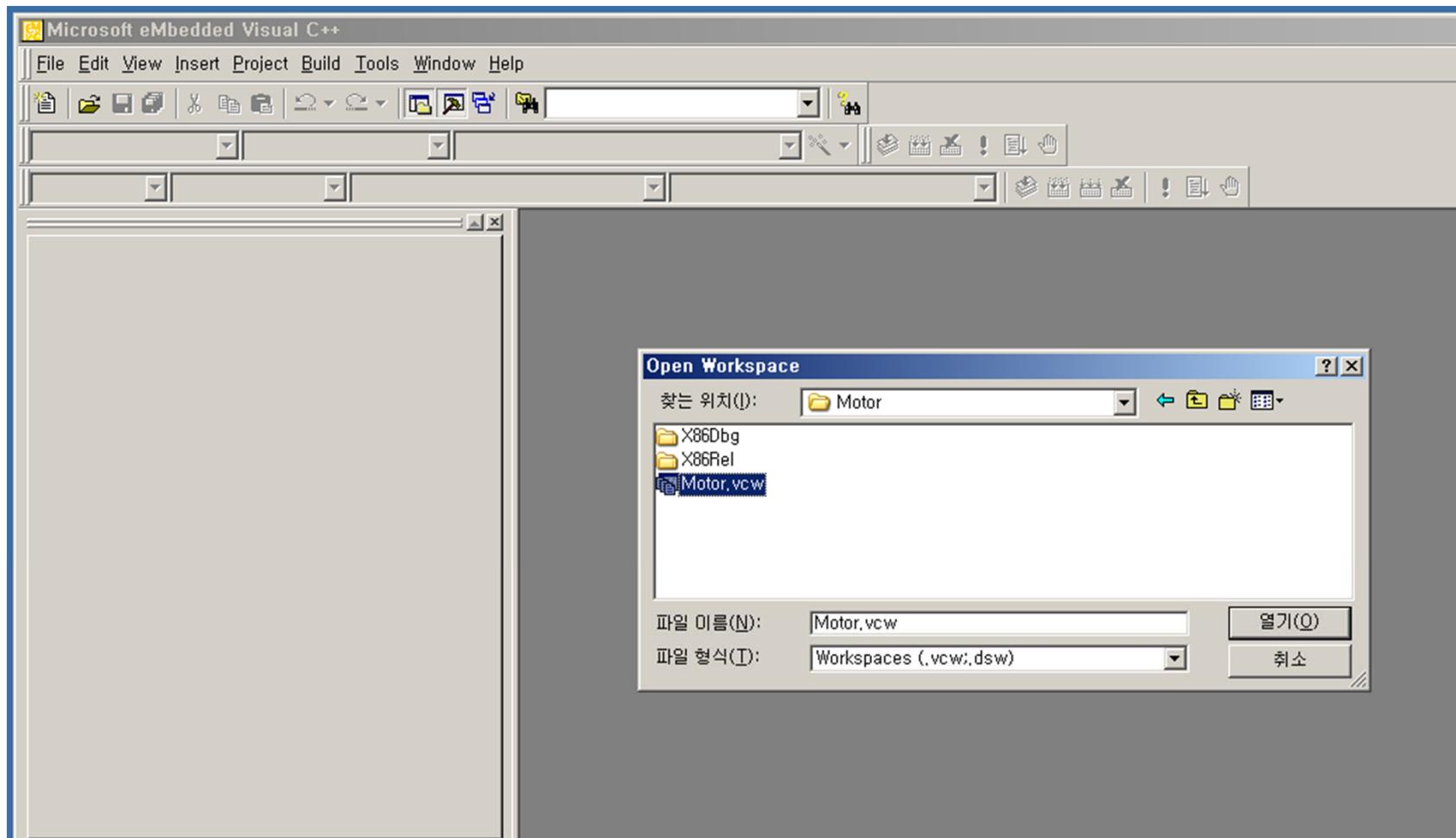
Find project workspace file



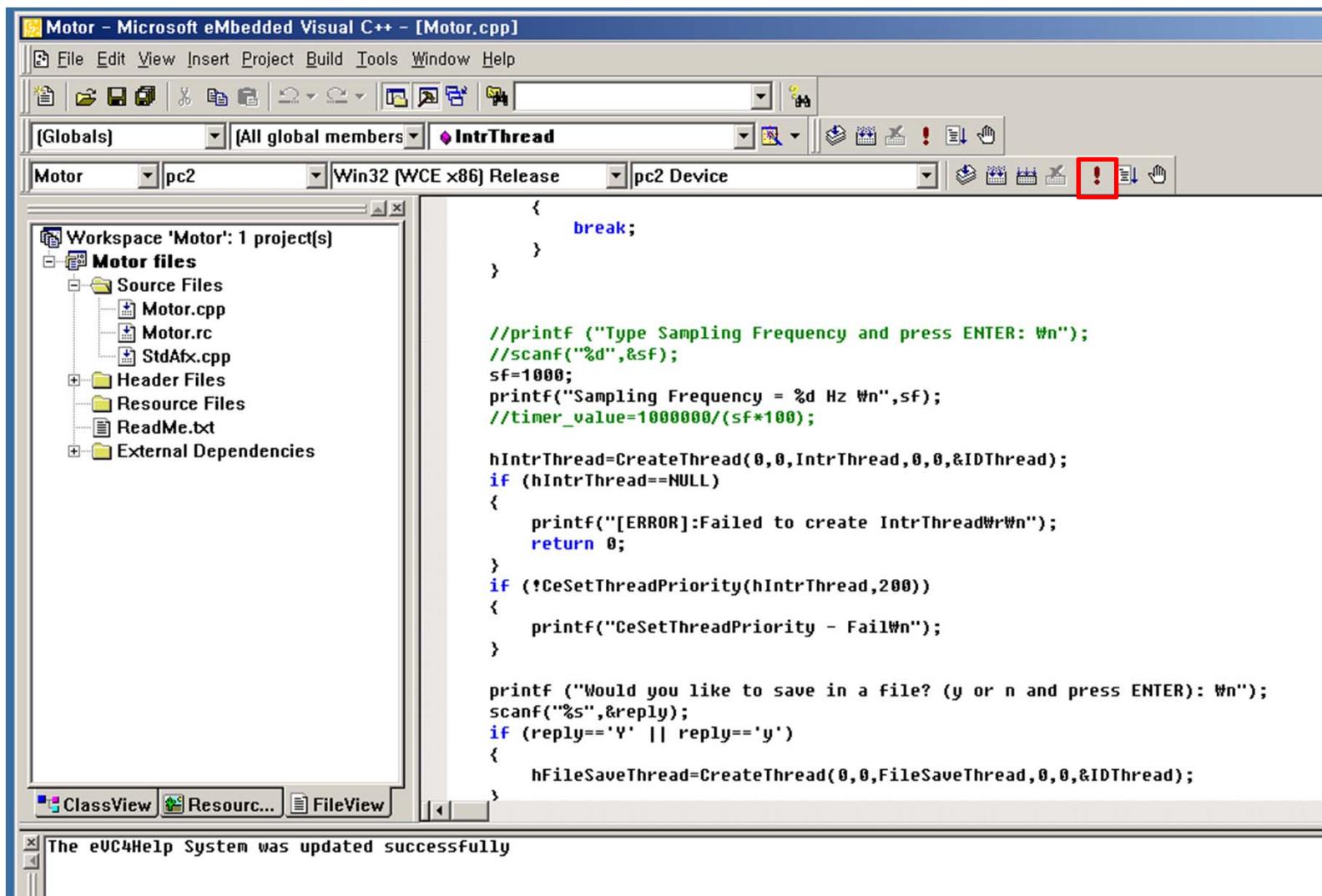
Open Workspace from File menu



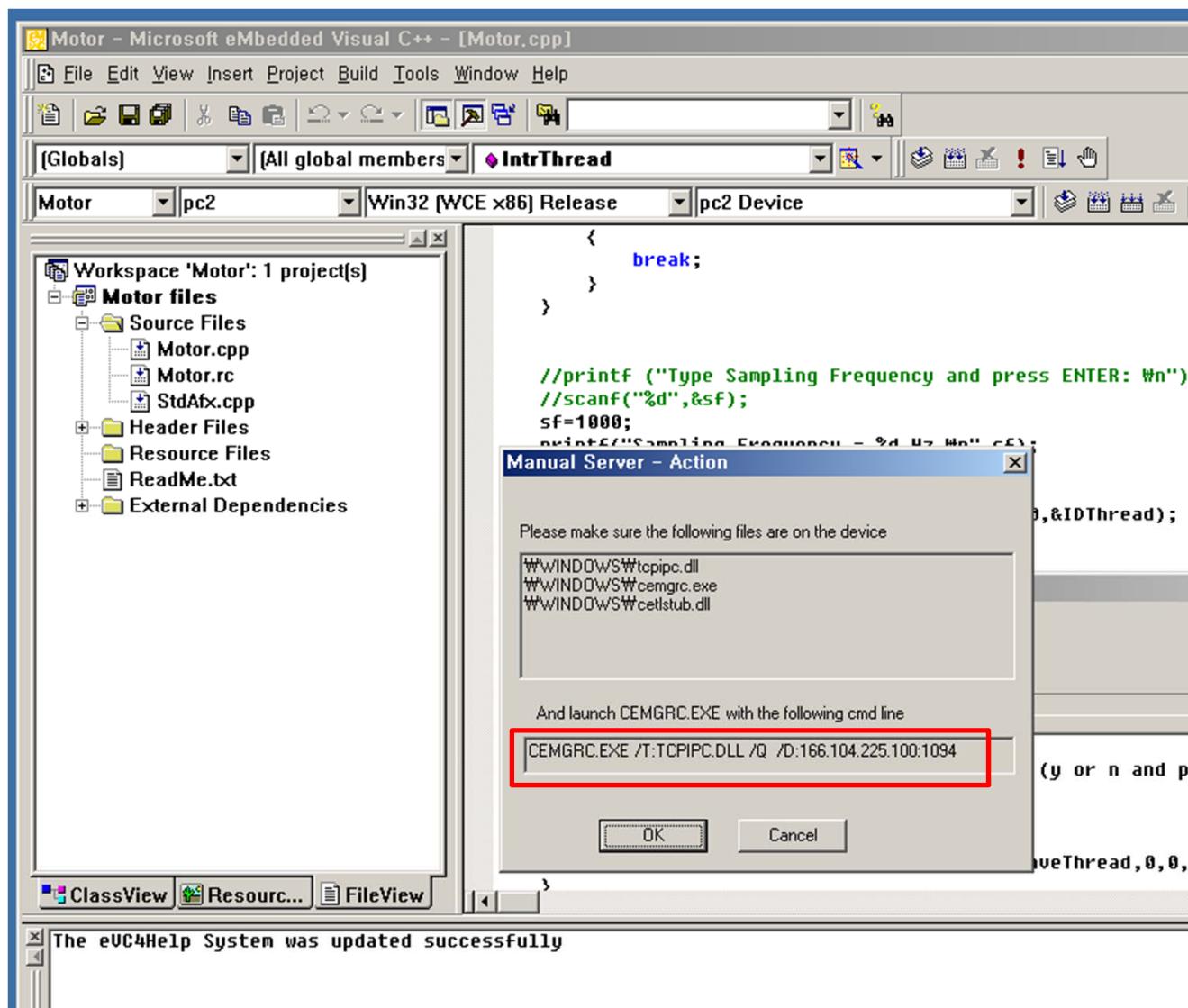
Open Workspace



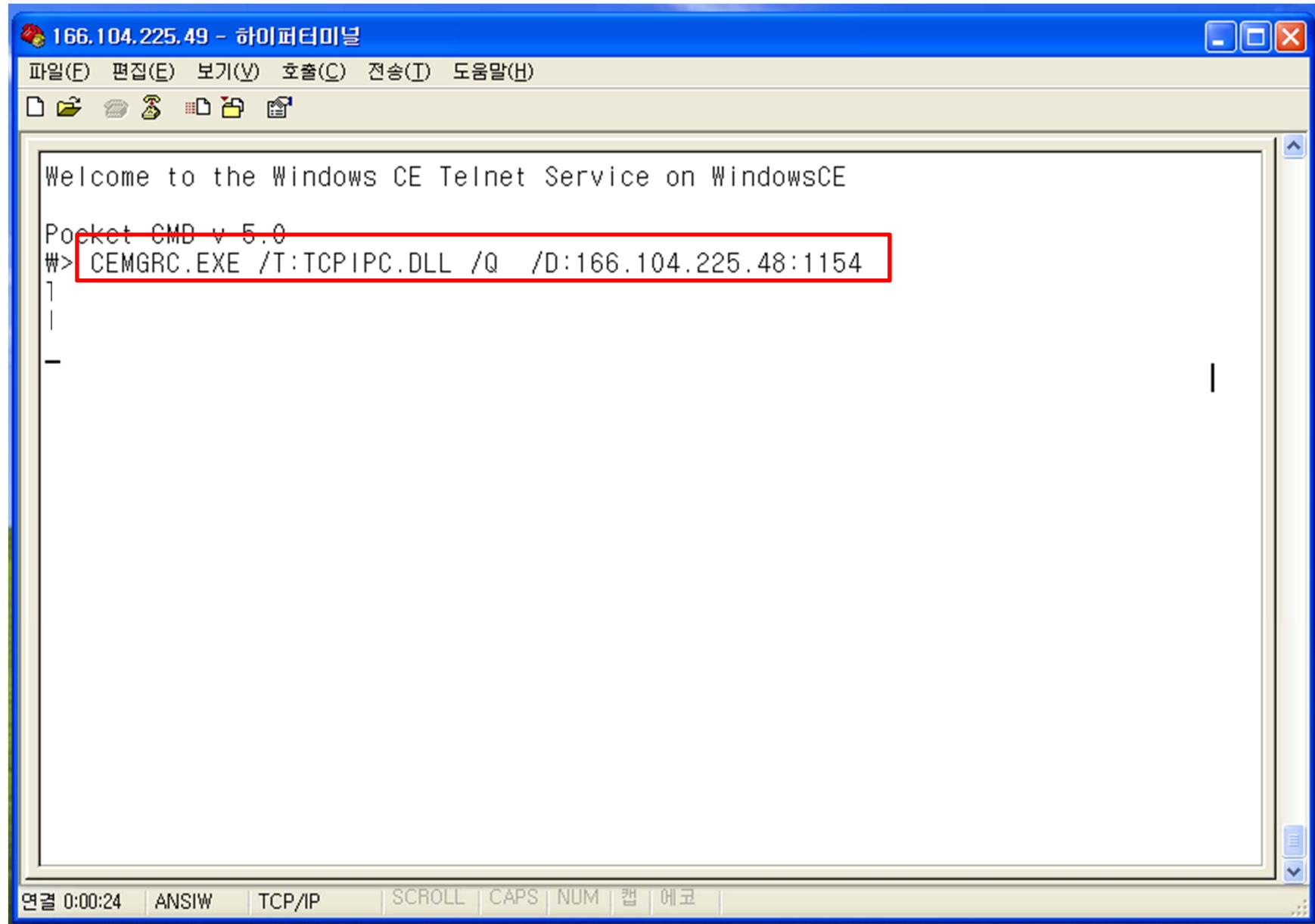
Press the execute button



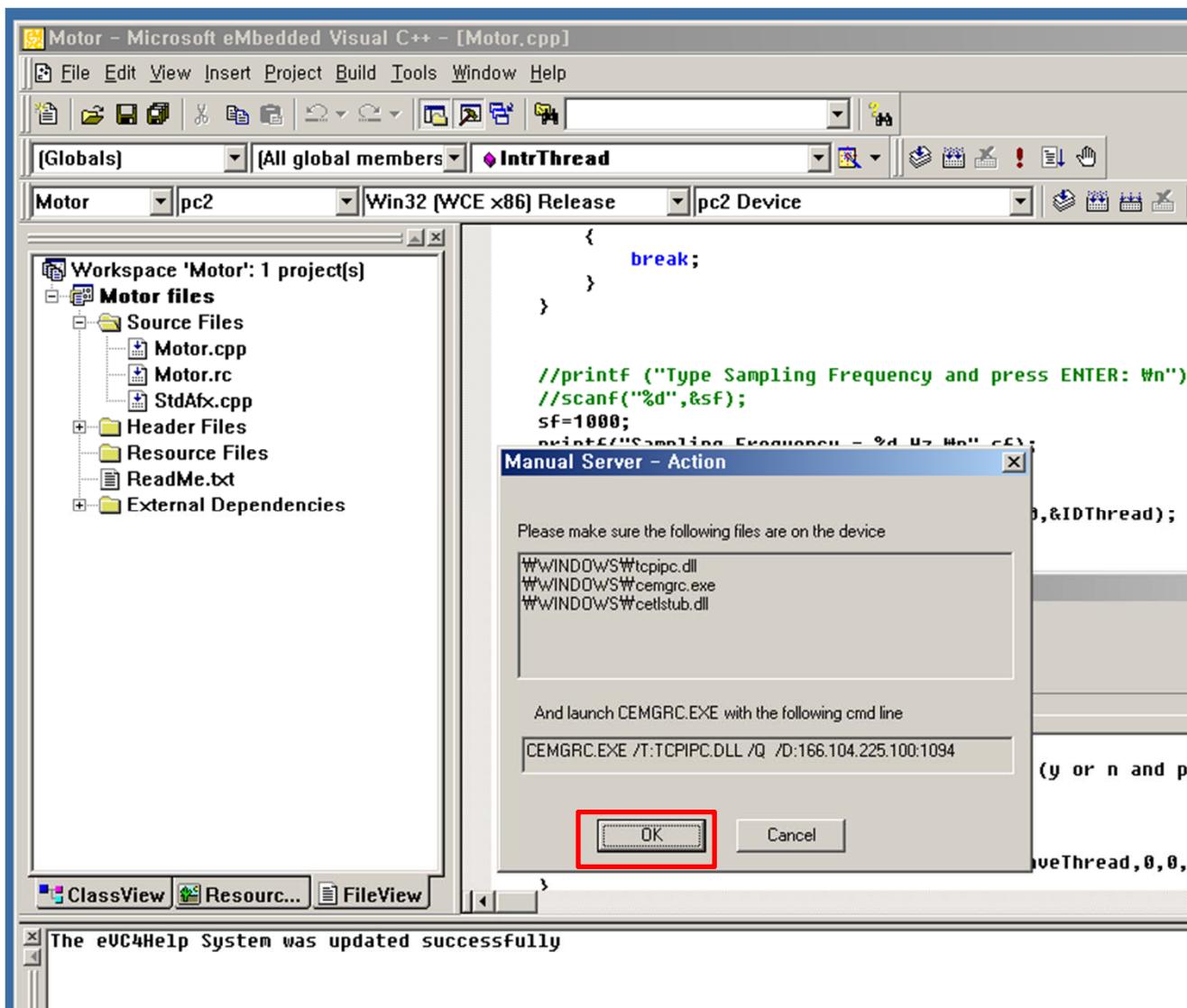
Copy the command

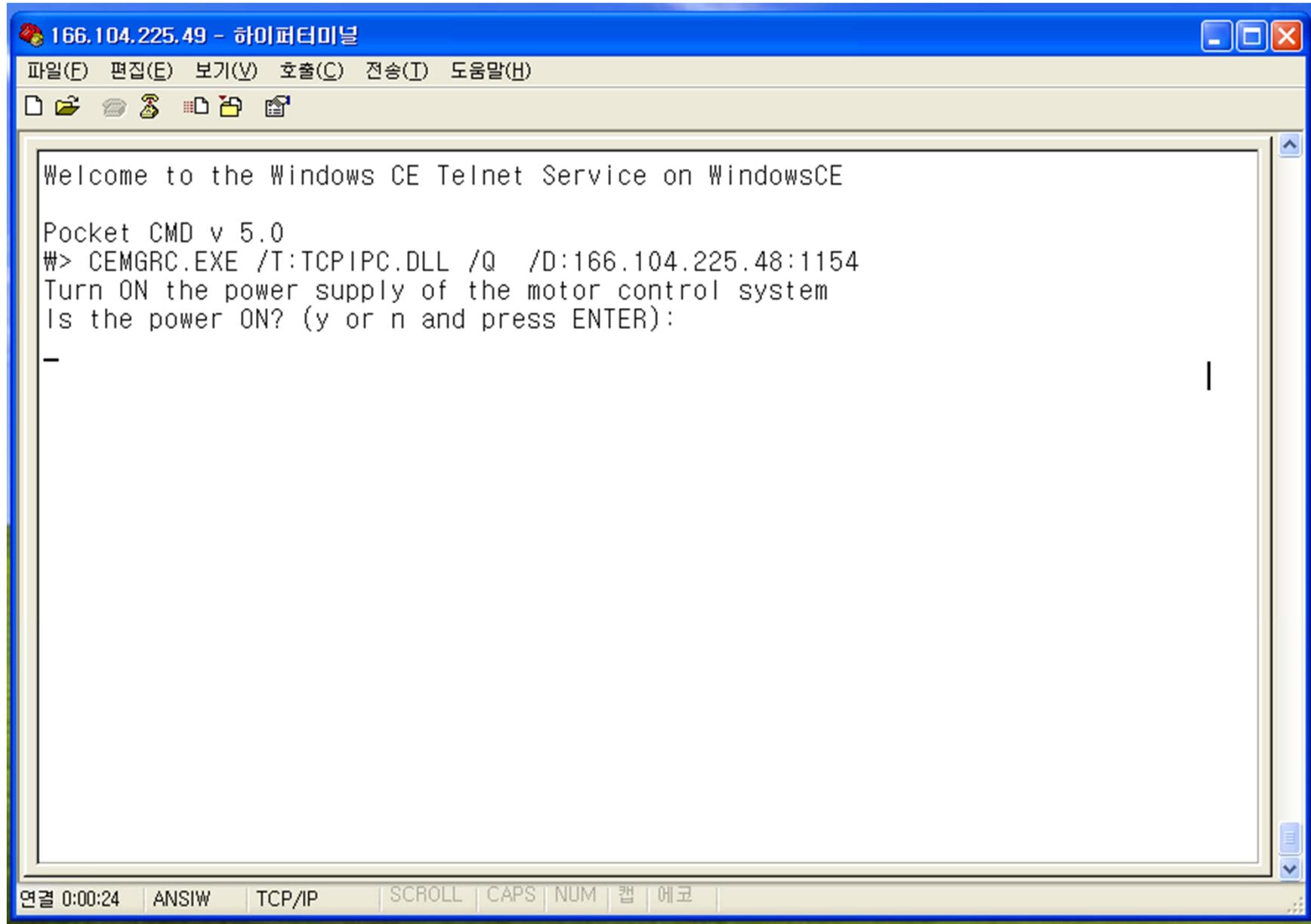


Paste and Enter on the Hyperterminal



Then, press OK





166.104.225.49 - 하이퍼터미널

파일(E) 편집(E) 보기(V) 호출(C) 전송(I) 도움말(H)



Welcome to the Windows CE Telnet Service on WindowsCE

Pocket CMD v 5.0

#> CEMGRC.EXE /T:TCPIPC.DLL /Q /D:166.104.225.48:1154

Turn ON the power supply of the motor control system

Is the power ON? (y or n and press ENTER):

y

Sampling Frequency = 1000 Hz

Interrupt Thread Initialize - Success

Would you like to save in a file? (y or n and press ENTER):

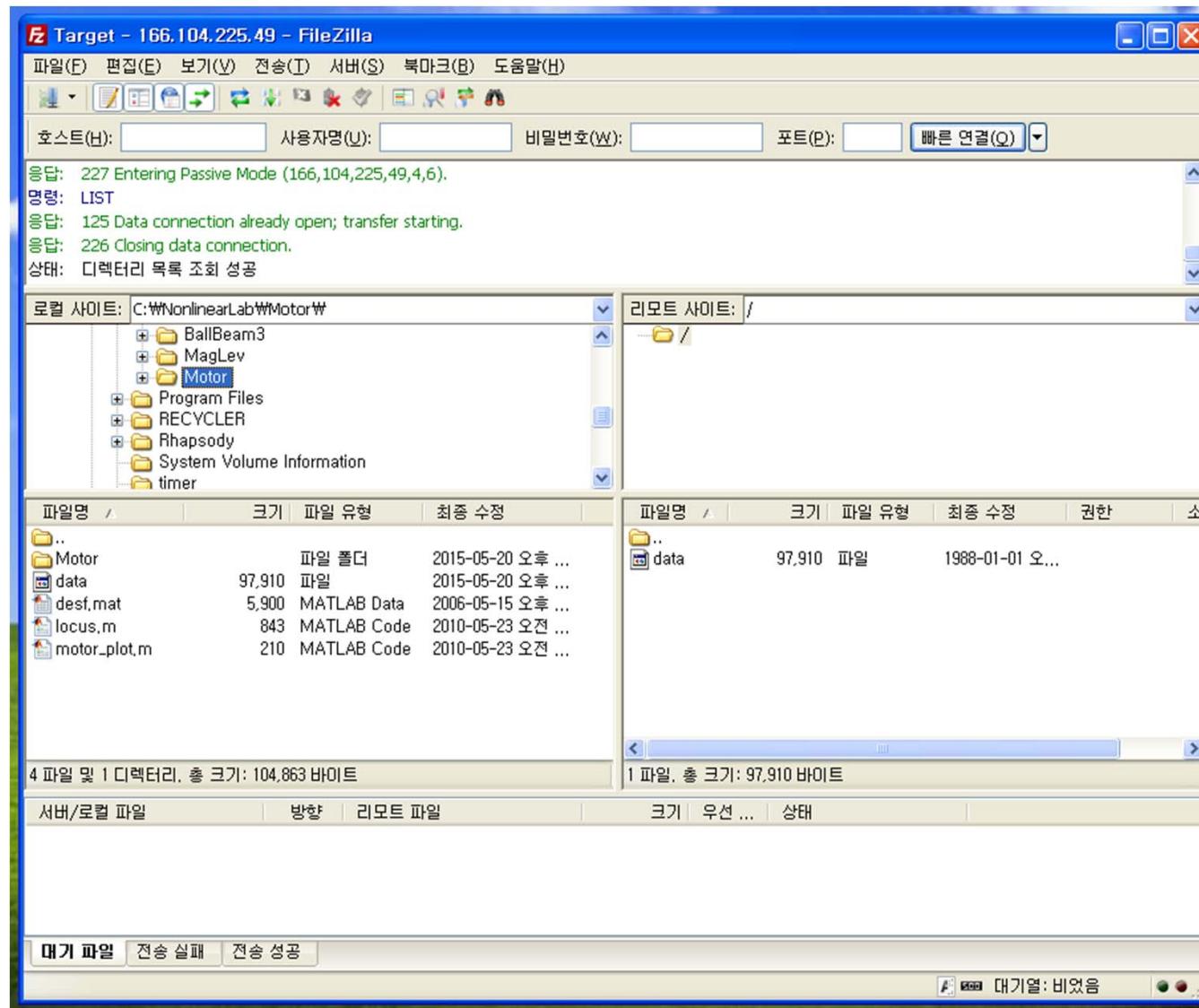
y

Data Capture Started

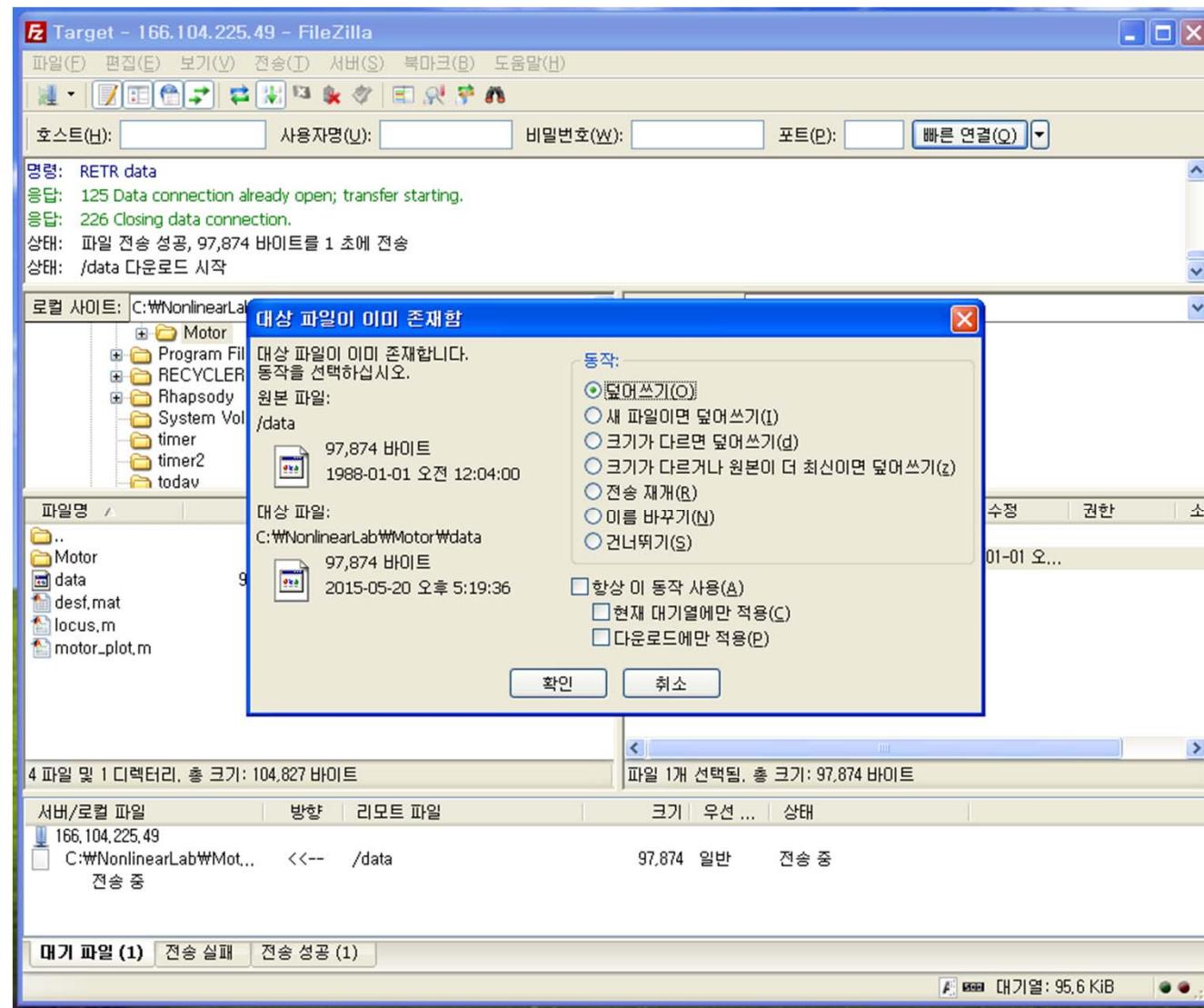
Data Capture Finished

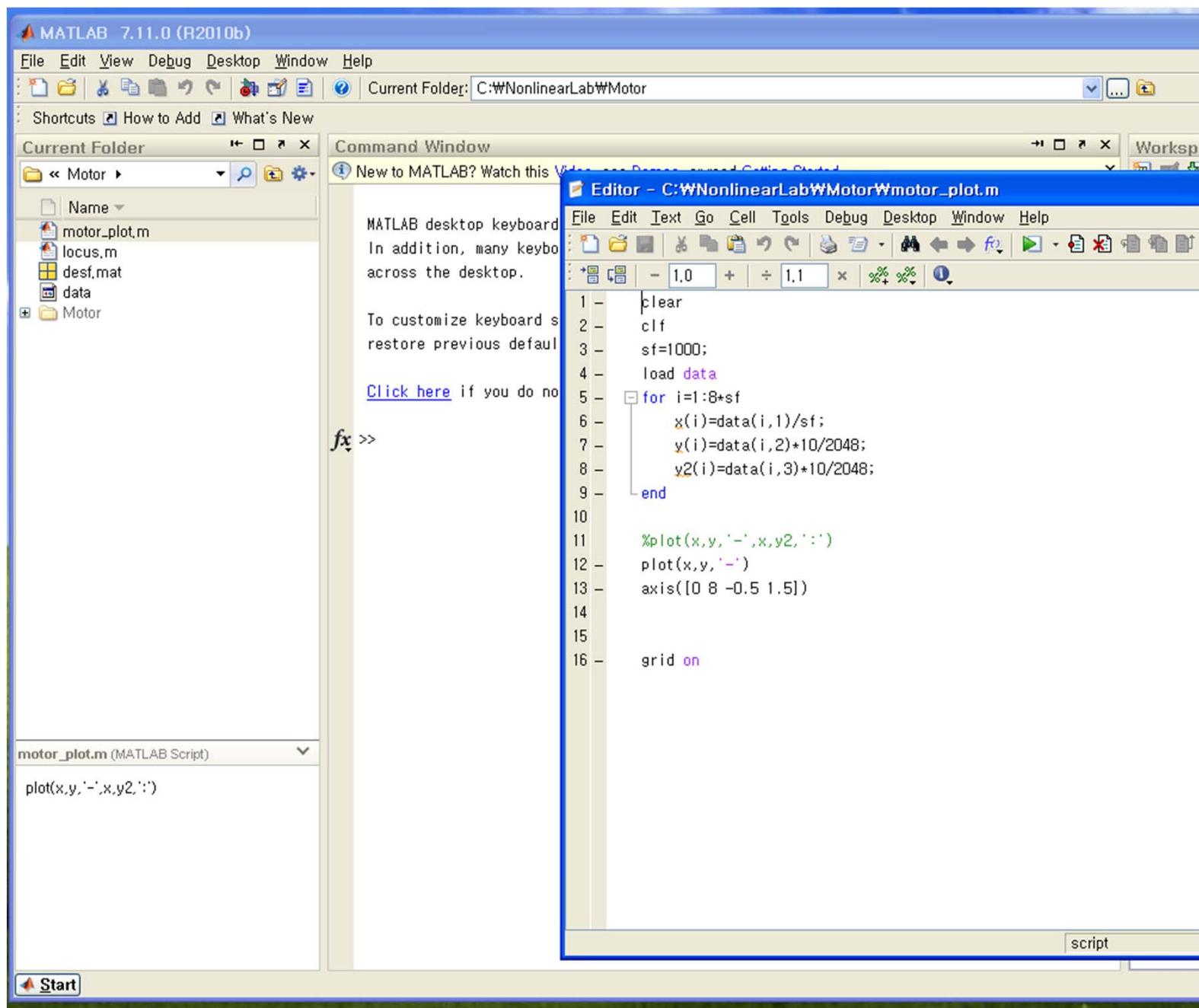
-

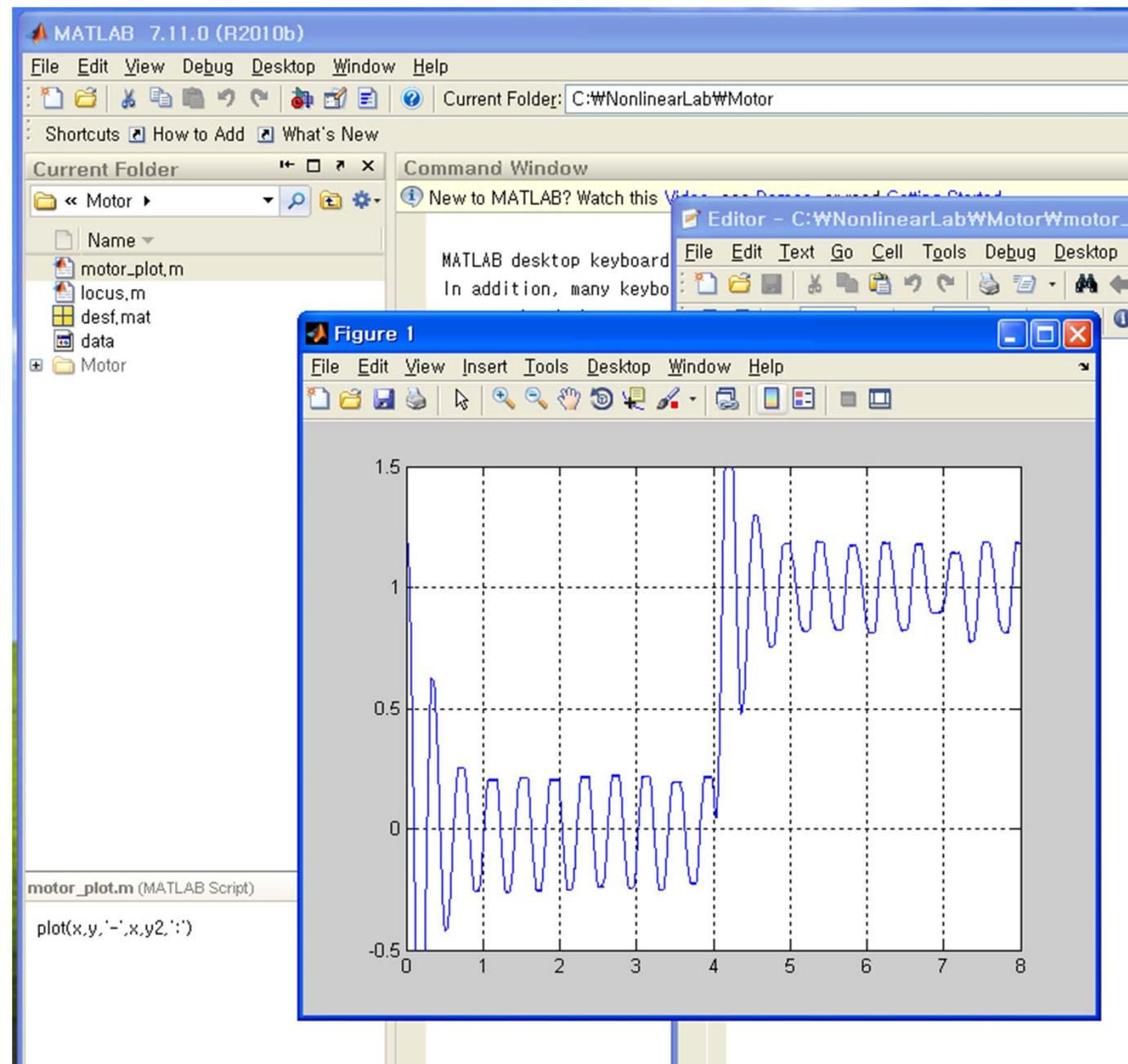
Double click data file on remote



Overwrite the file

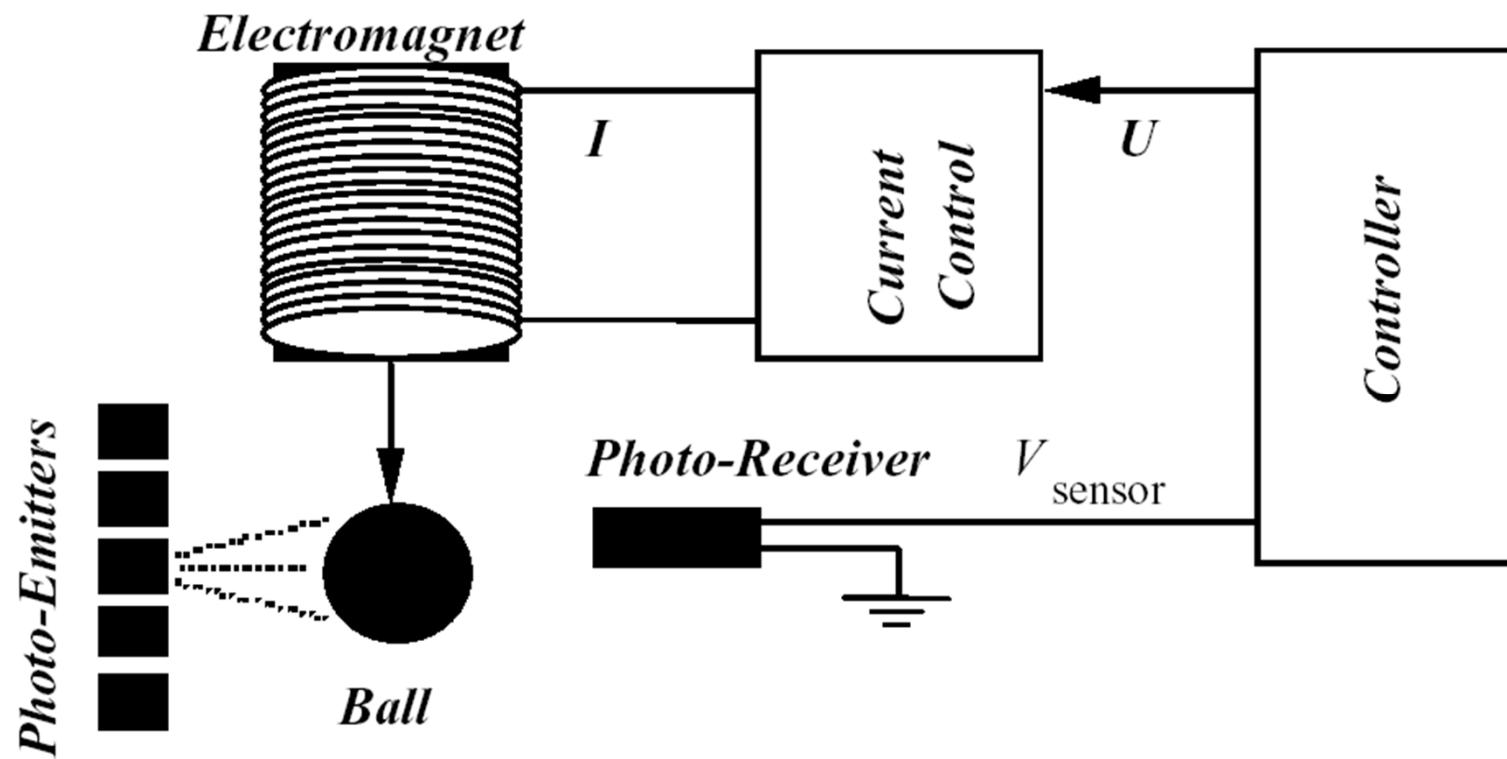






Magnetic Levitation

Magnetic Levitation System



Dynamic Equation

$$m\ddot{x} = mg - k \frac{i^2}{x^2}$$

m : ball mass

$$i = 0.15u + I_0$$

g : gravitational acceleration

$$v = \gamma(x - X_0)$$

γ : sensor gain

x : ball position from the magnet

i : input current for magnet

v : sensor output

u : voltage input for the current amplifier

$$k = \frac{mgX_0^2}{I_0^2}$$

State Equation

$$x_1 = v$$

$$x_2 = \dot{v} = \gamma \dot{x}$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} f_1(x_1, x_2, u) \\ f_2(x_1, x_2, u) \end{bmatrix} = \begin{bmatrix} x_2 \\ \gamma g - \frac{\gamma k}{m} \frac{(0.15u + I_0)^2}{(x_1/\gamma + X_0)^2} \end{bmatrix}$$

Lyapunov Linearization

$$x_1 = 0, x_2 = 0, u = 0$$

$$\begin{bmatrix} \Delta \dot{x}_1 \\ \Delta \dot{x}_2 \end{bmatrix} = A \begin{bmatrix} \Delta x_1 \\ \Delta x_2 \end{bmatrix} + B \Delta u$$

$$A = \left[\begin{array}{cc} \frac{\partial f_1}{\partial x_1} & \frac{\partial f_1}{\partial x_2} \\ \frac{\partial f_2}{\partial x_1} & \frac{\partial f_2}{\partial x_2} \end{array} \right] \Bigg|_{x_1 = 0, x_2 = 0, u = 0}$$

$$= \left[\begin{array}{cc} 0 & 1 \\ \frac{2k}{m} (0.15u + I_0^2)^2 (\frac{x_1}{\gamma} + X_0)^{-3} & 0 \end{array} \right] \Bigg|_{x_1 = 0, x_2 = 0, u = 0} = \left[\begin{array}{cc} 0 & 1 \\ \frac{2g}{X_0} & 0 \end{array} \right]$$

Lyapunov Linearization

$$B = \left[\begin{array}{c} \frac{\partial f_1}{\partial u} \\ \frac{\partial f_2}{\partial u} \end{array} \right] \Bigg|_{x_1=0, x_2=0, u=0}$$
$$= \left[\begin{array}{c} 0 \\ -\frac{0.3\gamma k}{m} \frac{(0.15u + I_0)}{(x_1/\gamma + X_0)^2} \end{array} \right] \Bigg|_{x_1=0, x_2=0, u=0} = \left[\begin{array}{c} 0 \\ -\frac{0.3\gamma g}{I_0} \end{array} \right]$$

$$G(s) = -\frac{\eta}{s^2 - \omega_0^2}$$
$$\eta = \frac{0.3\gamma g}{I_0}$$

$$= -\frac{1200}{s^2 - 852}$$
$$\omega_0 = \sqrt{\frac{2g}{X_0}}$$

Parameters

$$\gamma = 1000/3 \text{ V/m}$$

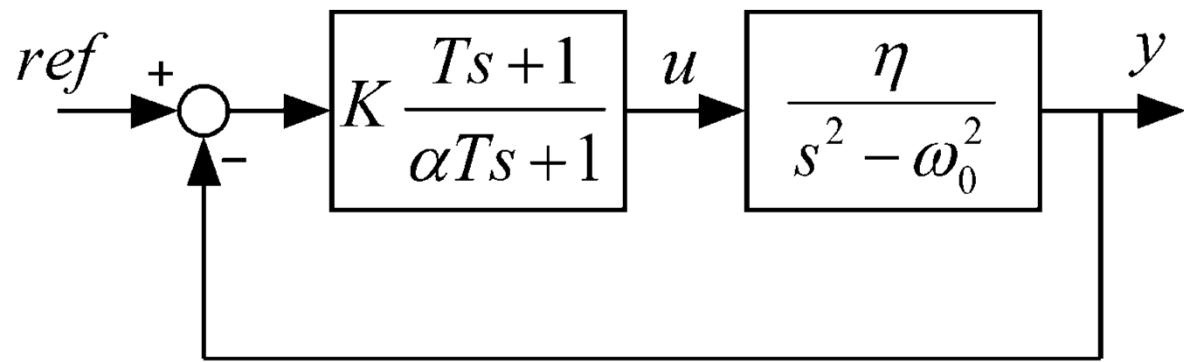
$$g = 9.8 \text{ m/sec}^2$$

$$I_0 = 0.817 \text{ A}$$

$$X_0 = 23/1000 \text{ m}$$

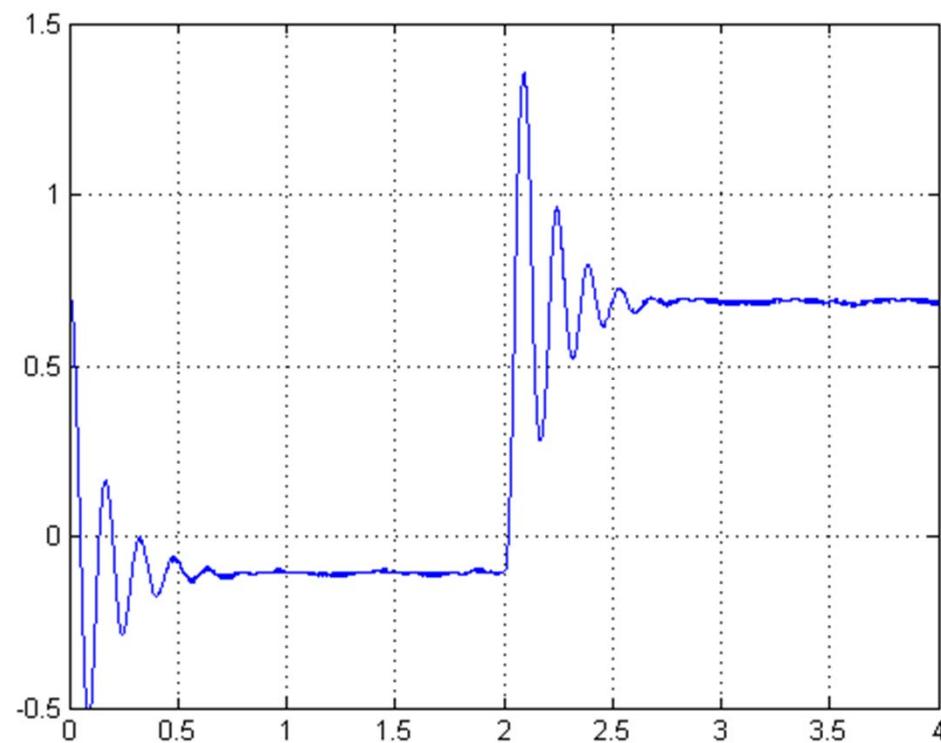
$$m = 21/1000 \text{ Kg}$$

Lead Compensator



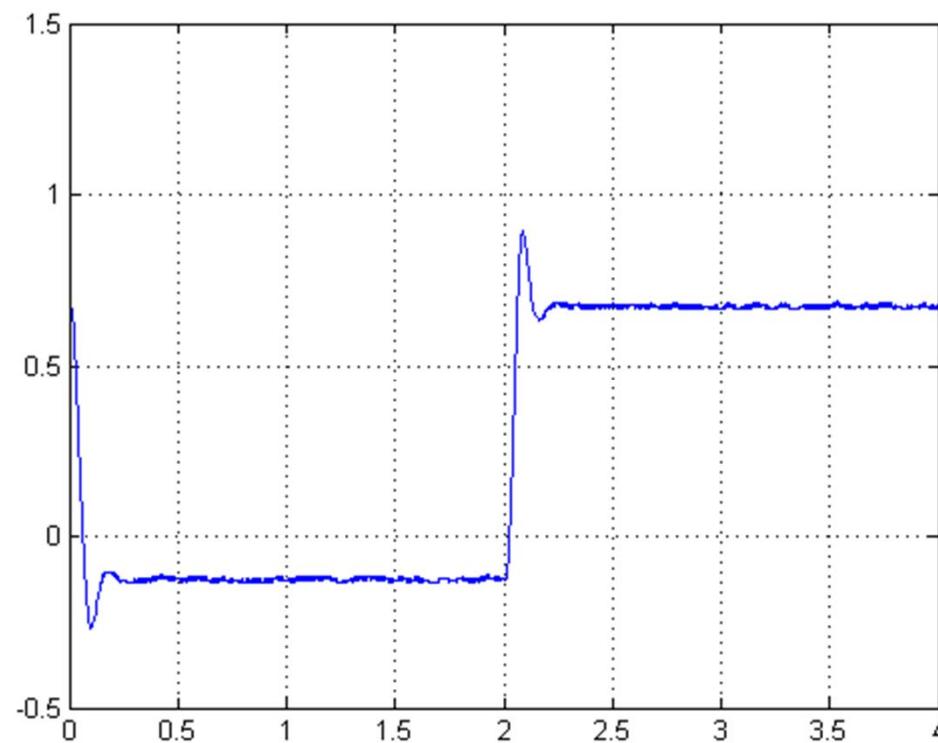
Lead Compensator

$$D_1(s) = 2 \frac{0.01s + 1}{0.001s + 1}$$



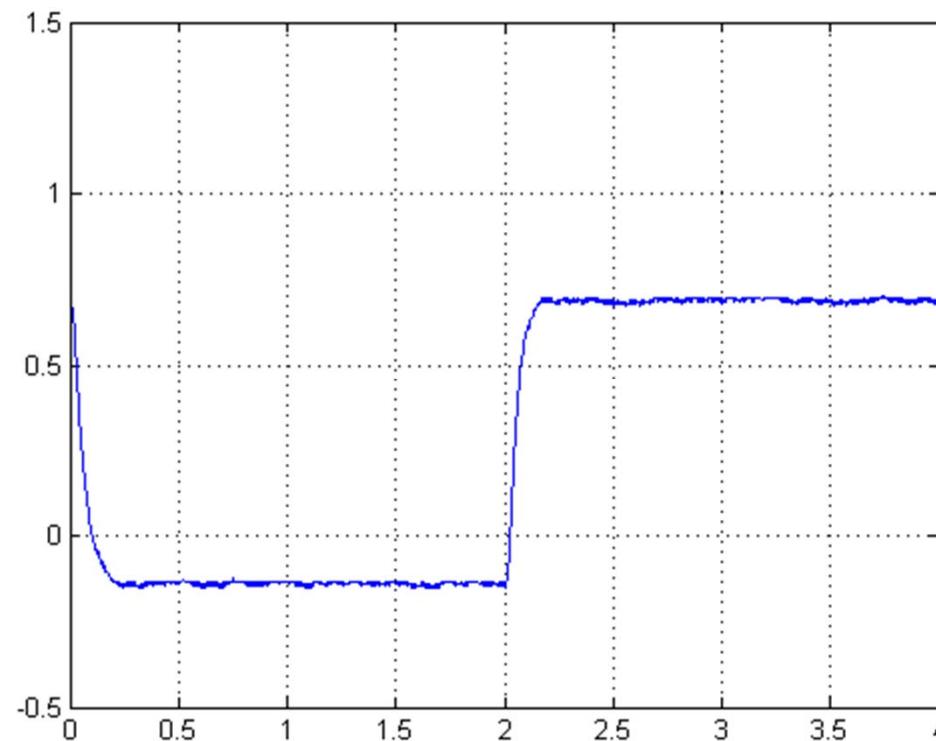
Lead Compensator

$$D_2(s) = 2 \frac{0.02s + 1}{0.001s + 1}$$



Lead Compensator

$$D_3(s) = 2 \frac{0.04s + 1}{0.001s + 1}$$



Input-Output Linearization

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} f_1(x_1, x_2, u) \\ f_2(x_1, x_2, u) \end{bmatrix} = \begin{bmatrix} x_2 \\ \gamma g - \frac{\gamma k}{m} \frac{(0.15u + I_0)^2}{(x_1 / \gamma + X_0)^2} \end{bmatrix}$$

$$w = \gamma g - \frac{\gamma k}{m} \frac{(0.15u + I_0)^2}{(x_1 / \gamma + X_0)^2}$$

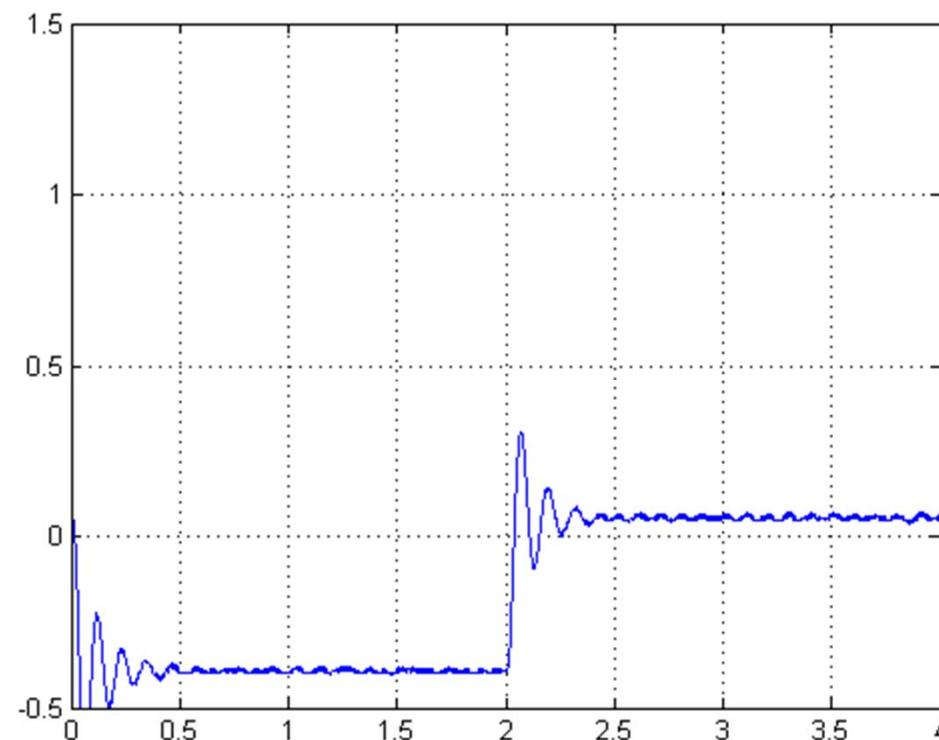
$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ w \end{bmatrix}$$

$$w = k_1(\text{ref} - x_1) - k_2 x_2$$

$$u = \left(\sqrt{(\gamma g - w)m(x_1 / \gamma + X_0)^2 / (\gamma k)} - I_0 \right) / 0.15$$

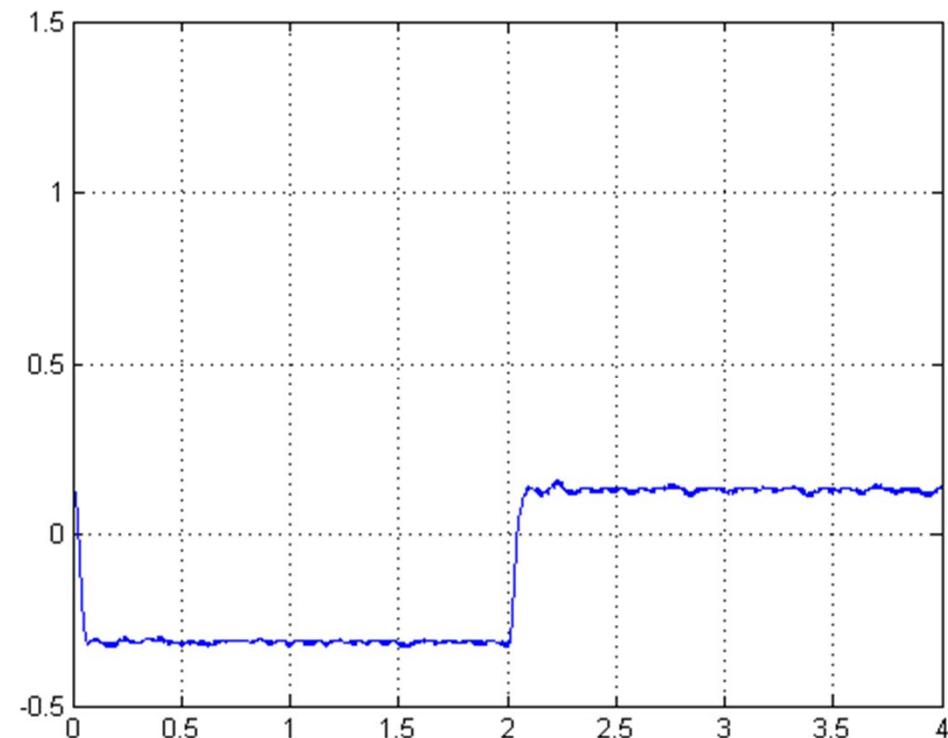
Input-Output Linearization

Poles: $-20 \pm j50$



Input-Output Linearization

Poles : -50, -50



Exercise 1

- Lead compensator에 대한 제어기를 실행하여 응답을 관찰하시오.
- 세 가지 lead compensator에 대해서, Bode plot을 그리고, step response의 simulation을 실행하여, phase margin의 변화에 따라서 응답이 어떻게 달라지는지 관찰하시오. 또한 simulation과 실험 결과를 비교해 보시오.

Exercise 1

- Observe the step responses with lead compensators.
- For the three lead compensators, plot Bode plots and step response simulations. Discuss how the phase margins affects the transients of responses. Also compare the step response simulations with the experiments.

Exercise 2

- Input-output linearization method를 이용한 제어기를 주어진 소스 코드를 활용하여 구현하시오. Linear controller는 closed-loop pole이 다음과 같은 state feedback controller로 구성하시오.

Poles : $-20 \pm j50$

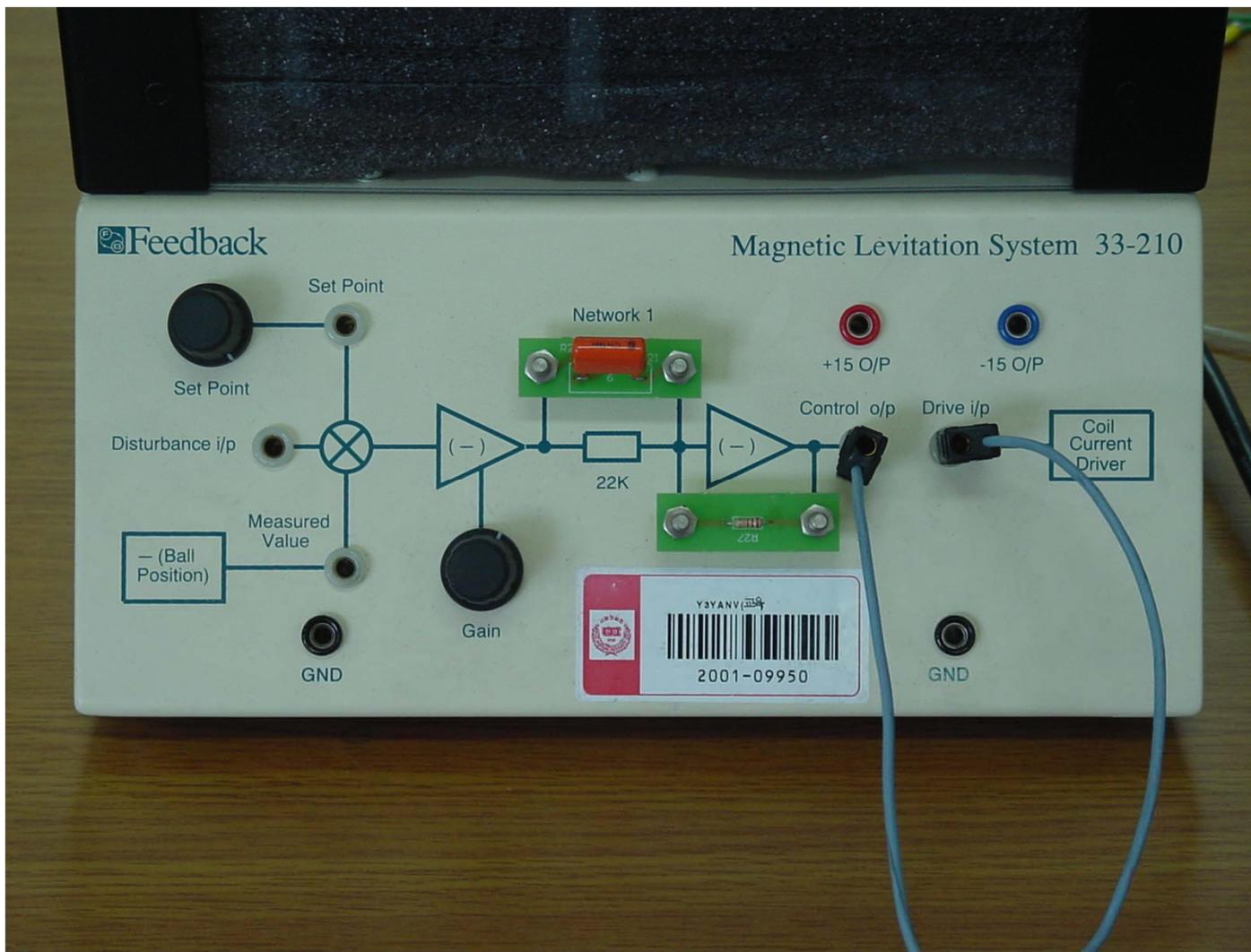
Poles : $-50, -50$

Exercise 2

- Implement the state feedback controller using the input-output linearization method. The closed-loop poles of the linear state feedback controller are as follows:

Poles : $-20 \pm j50$

Poles : $-50, -50$



Ball & Beam

IEEE Transactions on Automatic Control, March 1992

Nonlinear Control Via Approximate Input-Output Linearization: The Ball and Beam Example

John Hauser, Shankar Sastry, and Petar Kokotović

Abstract—We study approximate input-output linearization of non-linear systems which fail to have a well defined relative degree. For such systems, we provide a method for constructing approximate systems that are input-output linearizable. The analysis presented in this note is motivated through its application to a common undergraduate control laboratory experiment—the ball and beam system—where it is shown to be more effective for trajectory tracking than the standard Jacobian linearization.

Ball & Beam

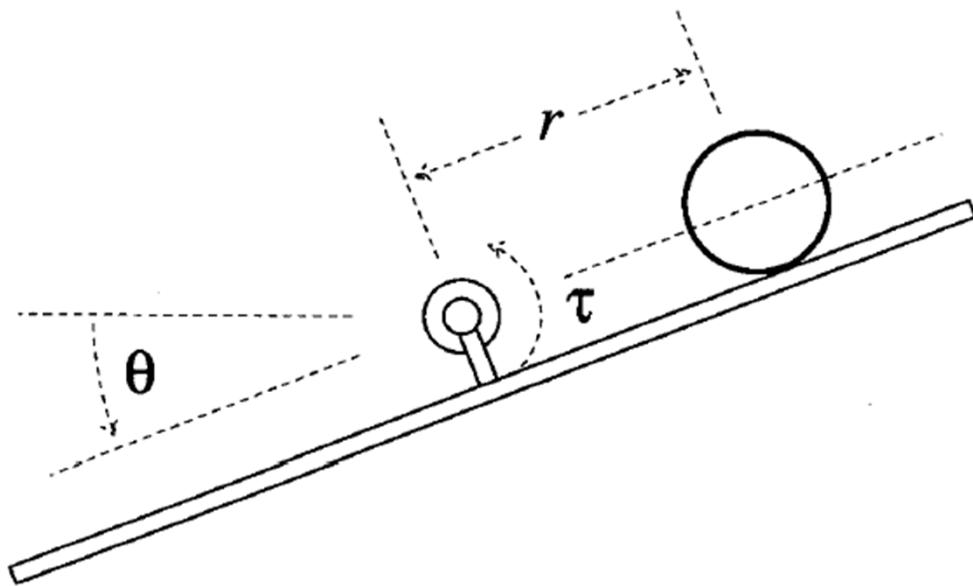


Fig. 1. The ball and beam system.

$$0 = \left(\frac{J_b}{R^2} + M \right) \ddot{r} + MG \sin \theta - Mr\dot{\theta}^2$$

$$\tau = (Mr^2 + J + J_b)\ddot{\theta} + 2Mr\dot{r}\dot{\theta} + MGr \cos \theta$$

State Equation

$$B = \frac{M}{\left(\frac{J_b}{R^2} + M \right)} \quad \tau = 2Mr\dot{r}\dot{\theta} + MGr \cos \theta + (Mr^2 + J + J_b)u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \underbrace{\begin{bmatrix} x_2 \\ B(x_1x_4^2 - G \sin x_3) \\ x_4 \\ 0 \end{bmatrix}}_{f(x)} + \underbrace{\begin{bmatrix} 0 \\ 0 \\ 0 \\ 1 \end{bmatrix}}_{g(x)} u$$

$$y = \underbrace{x_1}_{h(x)}$$

where $x = (x_1, x_2, x_3, x_4)^T := (r, \dot{r}, \theta, \dot{\theta})^T$ and $y = h(x) := r$.

Input-Output Linearization

$$y = x_1,$$

$$\dot{y} = x_2,$$

$$\ddot{y} = Bx_1x_4^2 - BG \sin x_3,$$

$$y^{(3)} = \underbrace{Bx_2x_4^2 - BGx_4 \cos x_3}_{b(x)} + \underbrace{2Bx_1x_4}_{{a}(x)} u.$$

$$u = (-b(\bar{x}) + v)/a(x) \quad y^{(3)} = v.$$

- Relative degree is not well defined!

Input-State Linearization

- Integrability condition is not satisfied.

$$\text{span} \{ g, ad_f g, \dots, ad_f^{n-2} g \} \quad (2.5)$$

since $[g, ad_f^2 g] = (2Bx_1 \ -2Bx_2 \ 0 \ 0)^T$ does not lie in (2.5). Here $ad_f^i g$ denotes the iterated *Lie bracket* $[f, \dots [f, g] \dots]$. Thus, it is not possible to fully linearize the ball and beam system.

Approximate Input-Output Linearization: 1

Let $\xi_1 = \phi_1(x) = h(x)$.

$$\dot{\xi}_1 = \underbrace{x_2}_{\text{---}}$$

$$\xi_2 = \phi_2(x)$$

$$\dot{\xi}_2 = \underbrace{-BG \sin x_3}_{\text{---}} + \underbrace{Bx_1 x_4^2}_{\text{---}}$$

$$\dot{\xi}_3 = \underbrace{-BGx_4 \cos x_3}_{\text{---}} \quad \begin{matrix} \xi_3 = \phi_3(x) \\ \psi_2(x) \end{matrix}$$

$$\xi_4 = \phi_4(x)$$

$$\dot{\xi}_4 = \underbrace{BGx_4^2 \sin x_3}_{b(x)} + \underbrace{(-BG \cos x_3) u}_{a(x)}$$

$$\dot{\xi}_1 = \xi_2$$

$$\dot{\xi}_2 = \xi_3 + \psi_2(x)$$

$$\text{or} \quad \dot{\xi}_3 = \xi_4$$

$$\dot{\xi}_4 = b(x) + a(x)u$$

$$=: v(x, u).$$

(3.2)

Approximate Input-Output Linearization: 2

Let $\xi_1 = \phi_1(x) = h(x)$.

$$\dot{\xi}_1 = \underbrace{x_2}_{\xi_2 = \phi_2(x)}$$

$$\dot{\xi}_2 = \underbrace{-BG \sin x_3 + Bx_1 x_4^2}_{\xi_3 = \phi_3(x)}$$

$$\dot{\xi}_3 = \underbrace{-BGx_4 \cos x_3 + Bx_2 x_4^2}_{\xi_4 = \phi_4(x)} + \underbrace{2Bx_1 x_4 u}_{\psi_3(x, u)}$$

$$\dot{\xi}_4 = \underbrace{B^2 x_1 x_4^4 + B(1 - B)x_4^2 \sin x_3}_{b(x)} + \underbrace{(-BG \cos x_3 + 2Bx_2 x_4)u}_{a(x)}$$

$$\dot{\xi}_1 = \xi_2$$

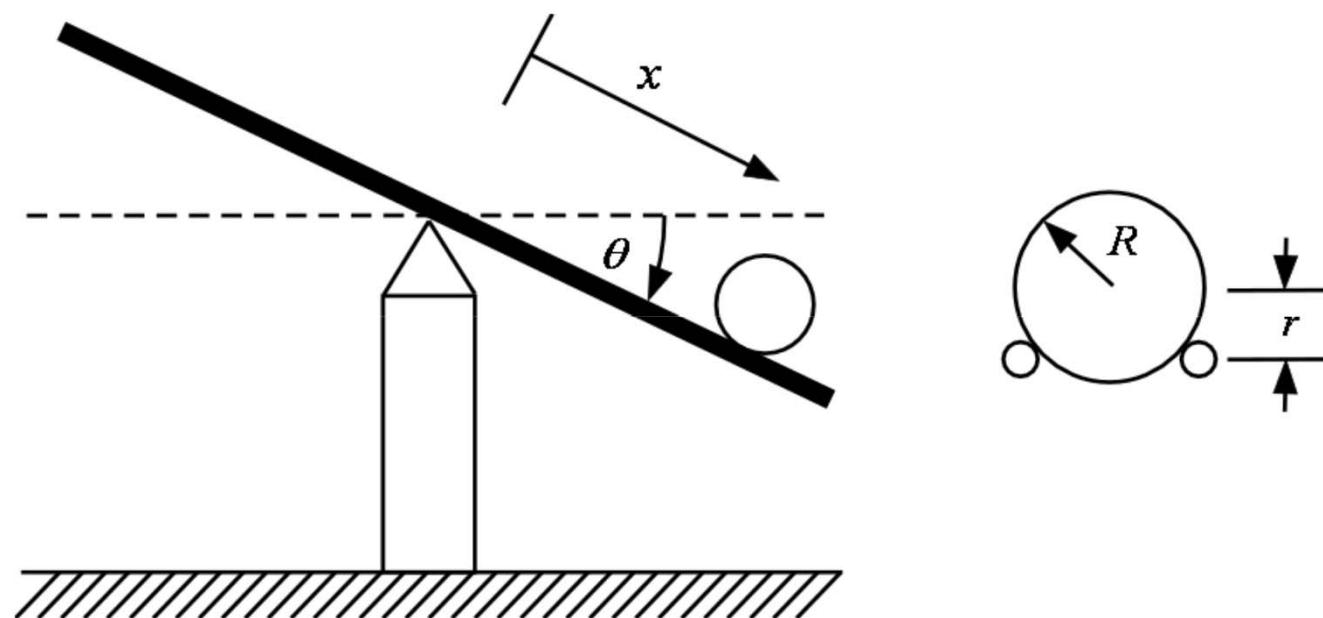
$$\dot{\xi}_2 = \xi_3$$

$$\begin{aligned} \text{or } \dot{\xi}_3 &= \xi_4 + \psi_3(x, u) \\ \dot{\xi}_4 &= b(x) + a(x)u \\ &=: v(x, u). \end{aligned}$$

$$v = y_d^{(4)}(t) + \alpha_3(y_d^{(3)}(t) - \phi_4(x)) + \alpha_2(\ddot{y}_d(t) - \phi_3(x))$$

$$+ \alpha_1(\dot{y}_d(t) - \phi_2(x)) + \alpha_0(y_d(t) - \phi_1(x)) \quad (3.1)$$

Ball & Beam



Dynamic Equation

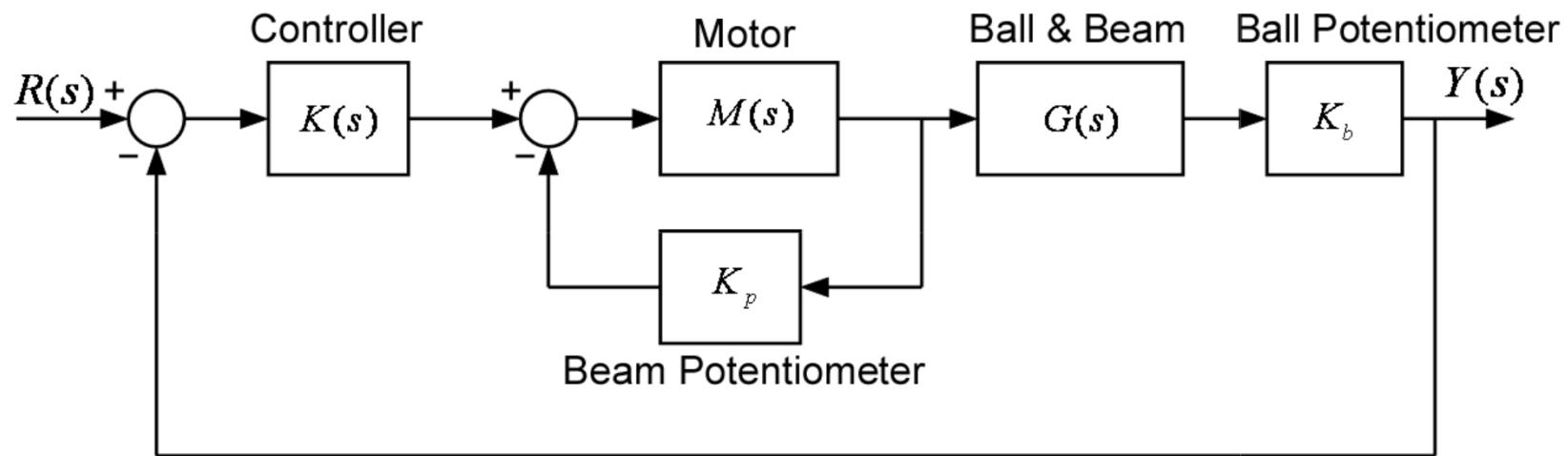
$$\left(m + \frac{I_b}{r^2} \right) \ddot{x} + \left(\frac{I_b}{r^2} \right) \ddot{\theta} - mx\dot{\theta}^2 = mg \sin \theta$$

Assume $\dot{\theta} \approx 0, \ddot{\theta} \approx 0, \sin \theta \approx \theta$

$$\left(m + \frac{I_b}{r^2} \right) \ddot{x} = mg\theta, I_b = \frac{2}{5}mR^2$$

$$\frac{X(s)}{\Theta(s)} = \frac{mg}{\left(m + \frac{I_b}{r^2} \right) s^2} = \frac{g}{\left(1 + \frac{2}{5} \frac{R^2}{r^2} \right) s^2}$$

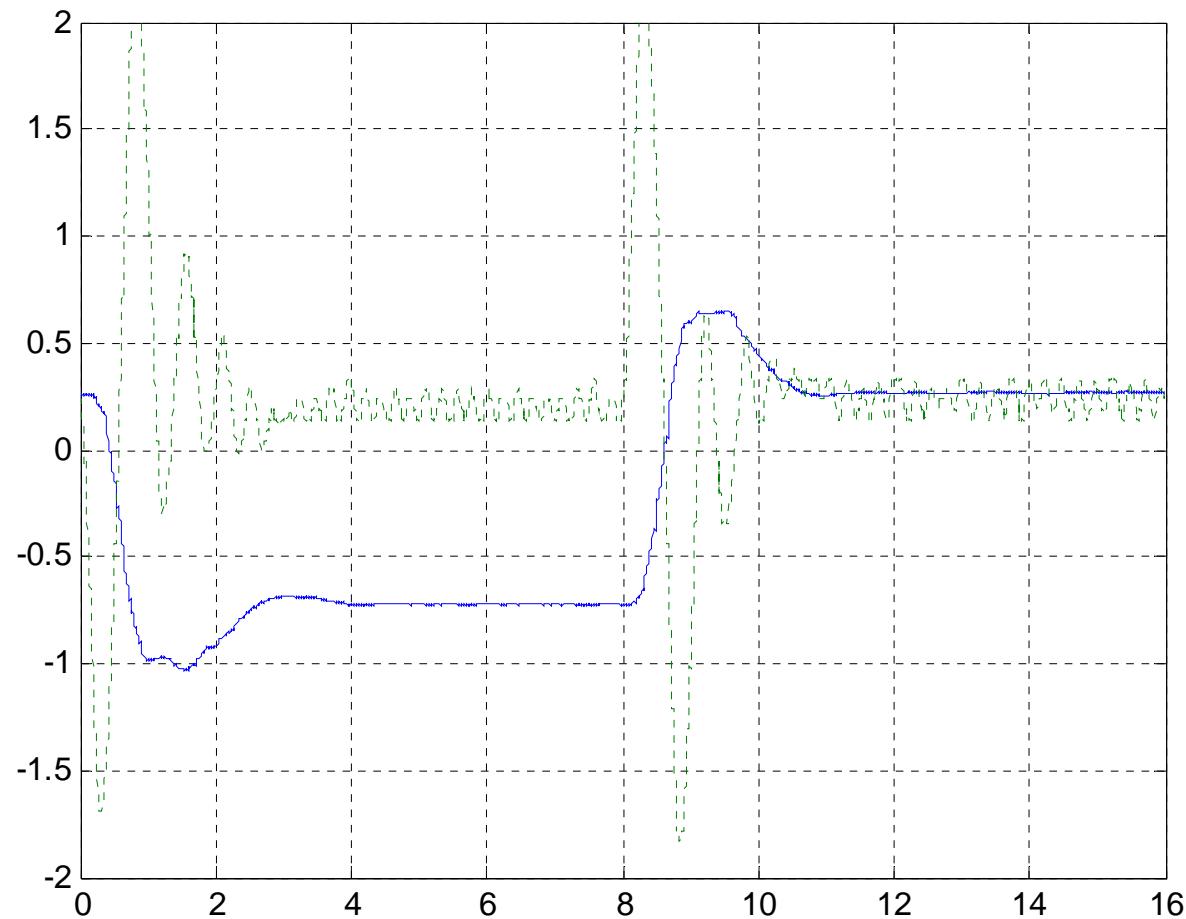
Control System



$$\frac{M(s)}{1 + K_p M(s)} \approx \frac{M(0)}{1 + K_p M(0)}$$

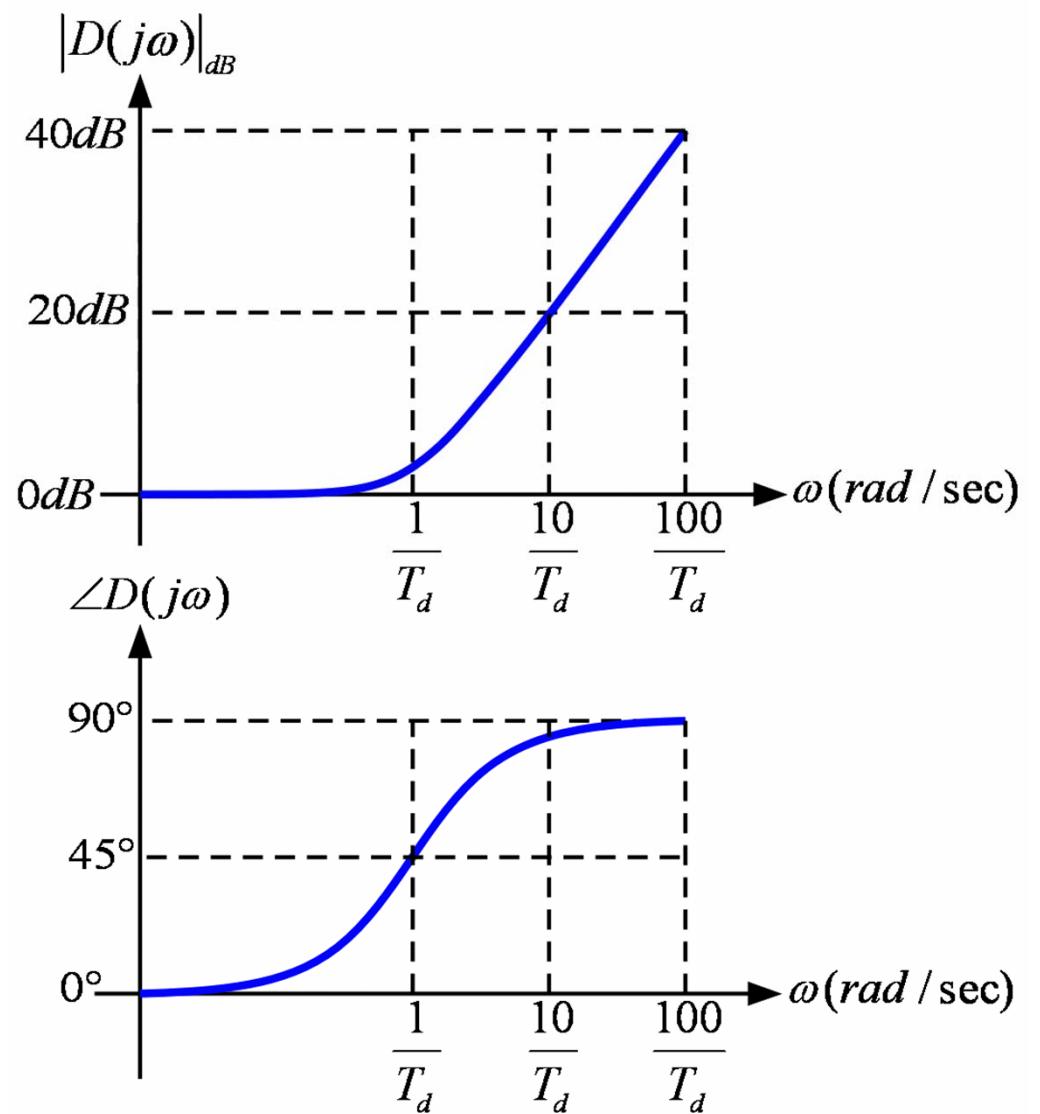
Exercise

- Try with PD controller.
- Design and implement two different lead compensator with $K=1$ and phase margin=30~50 degrees.
- Compare experimental results with simulations.



PD 제어기의 설계

$$D(s) = K(1 + T_d s)$$



예제 7-1

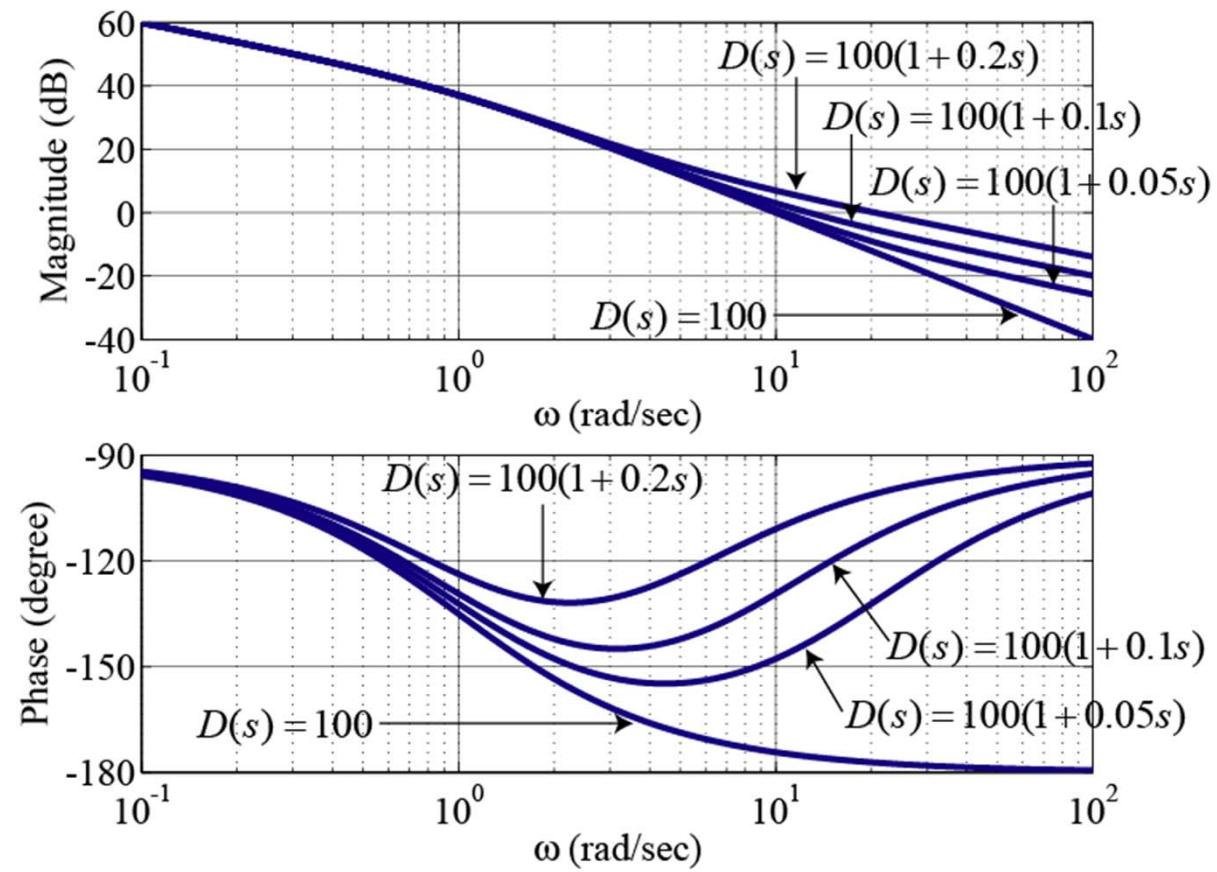
$$G(s) = \frac{1}{s(s+1)}$$

$$K_v = \lim_{s \rightarrow 0} s \frac{K}{s(s+1)} = K$$

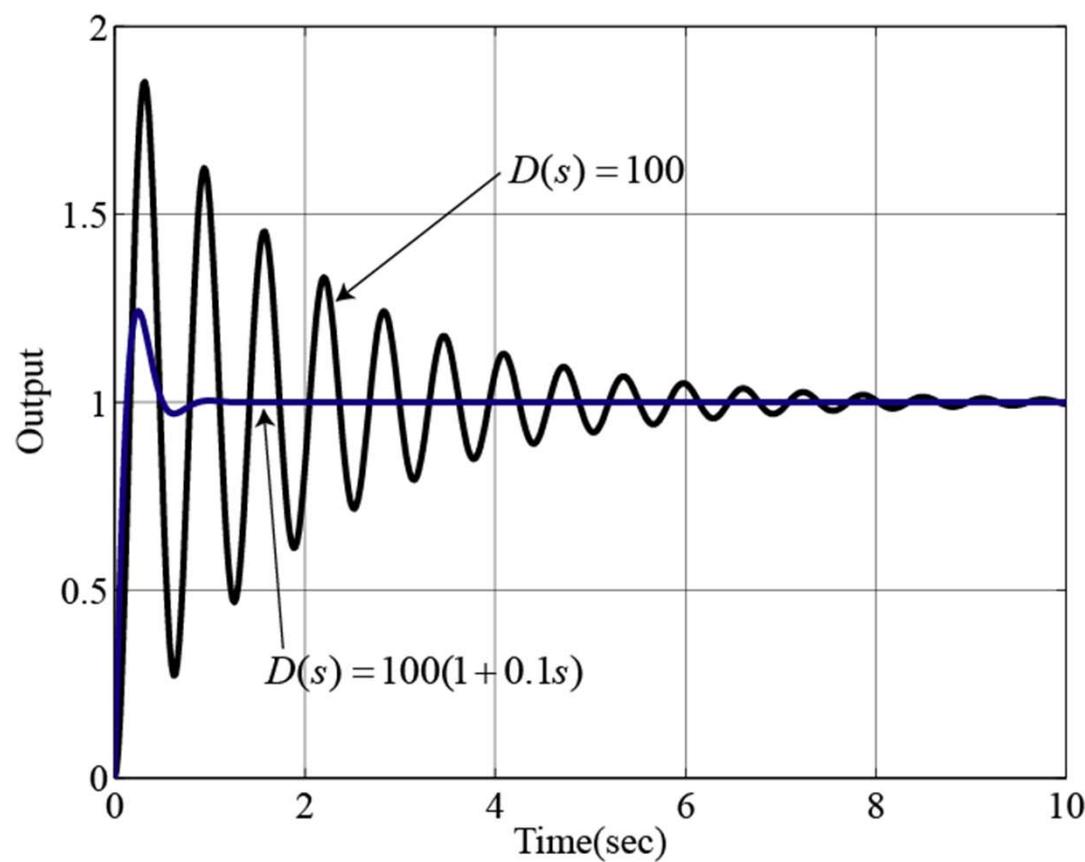
$$e_{ss} = \frac{1}{K_v} = \frac{1}{K}$$

$$D(s) = 100(1 + 0.1s)$$

예제 7-1

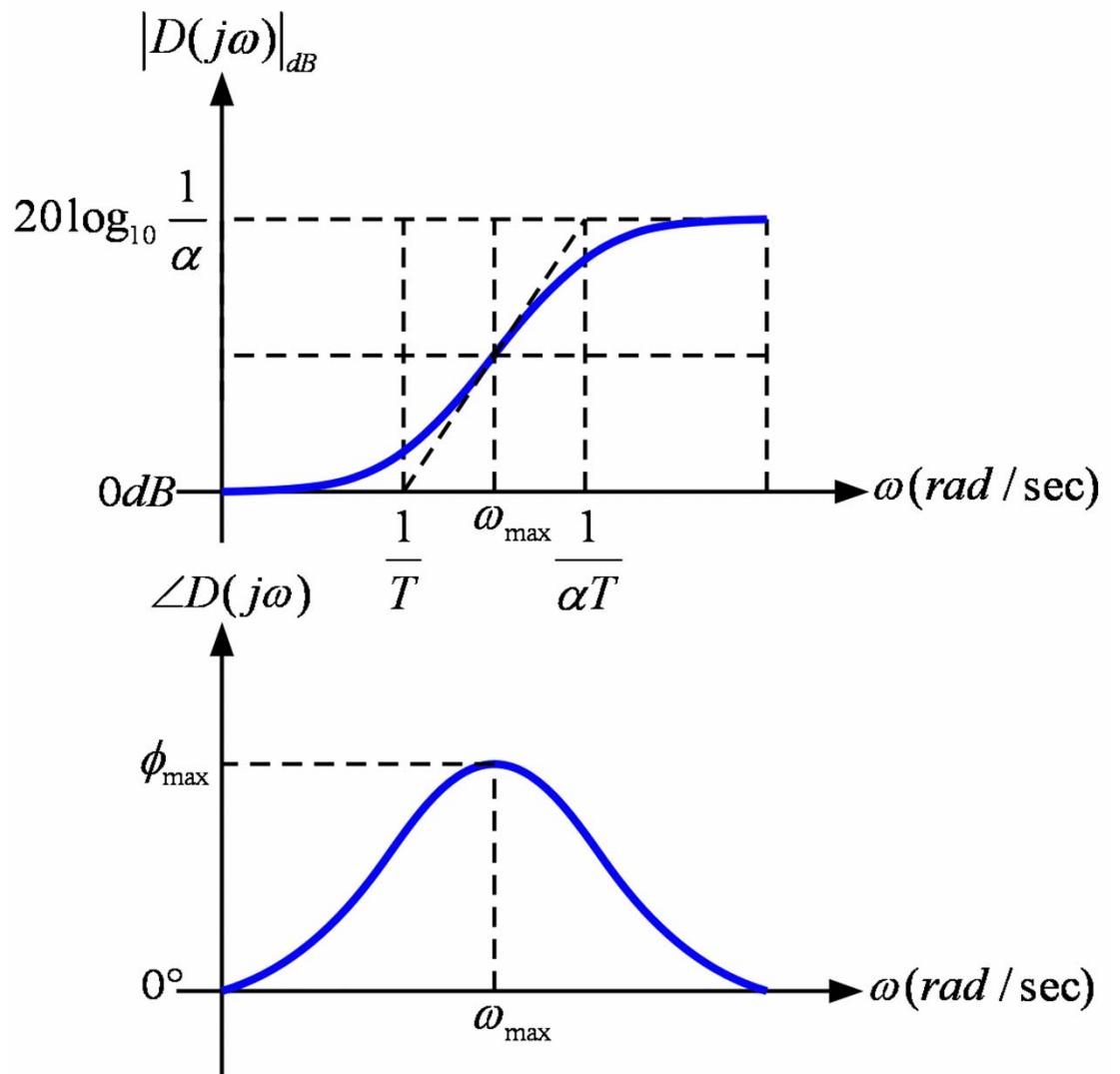


예제 7-1



진상 제어기의 설계

$$D(s) = K \frac{Ts + 1}{\alpha Ts + 1}$$



진상 제어기의 설계

$$\phi = \angle \left(\frac{jT\omega + 1}{j\alpha T\omega + 1} \right) = \tan^{-1}(T\omega) - \tan^{-1}(\alpha T\omega)$$

$$\log_{10} \omega_{\max} = \frac{1}{2} \left(\log_{10} \frac{1}{T} + \log_{10} \frac{1}{\alpha T} \right) \quad \omega_{\max} = \frac{1}{T\sqrt{\alpha}}$$

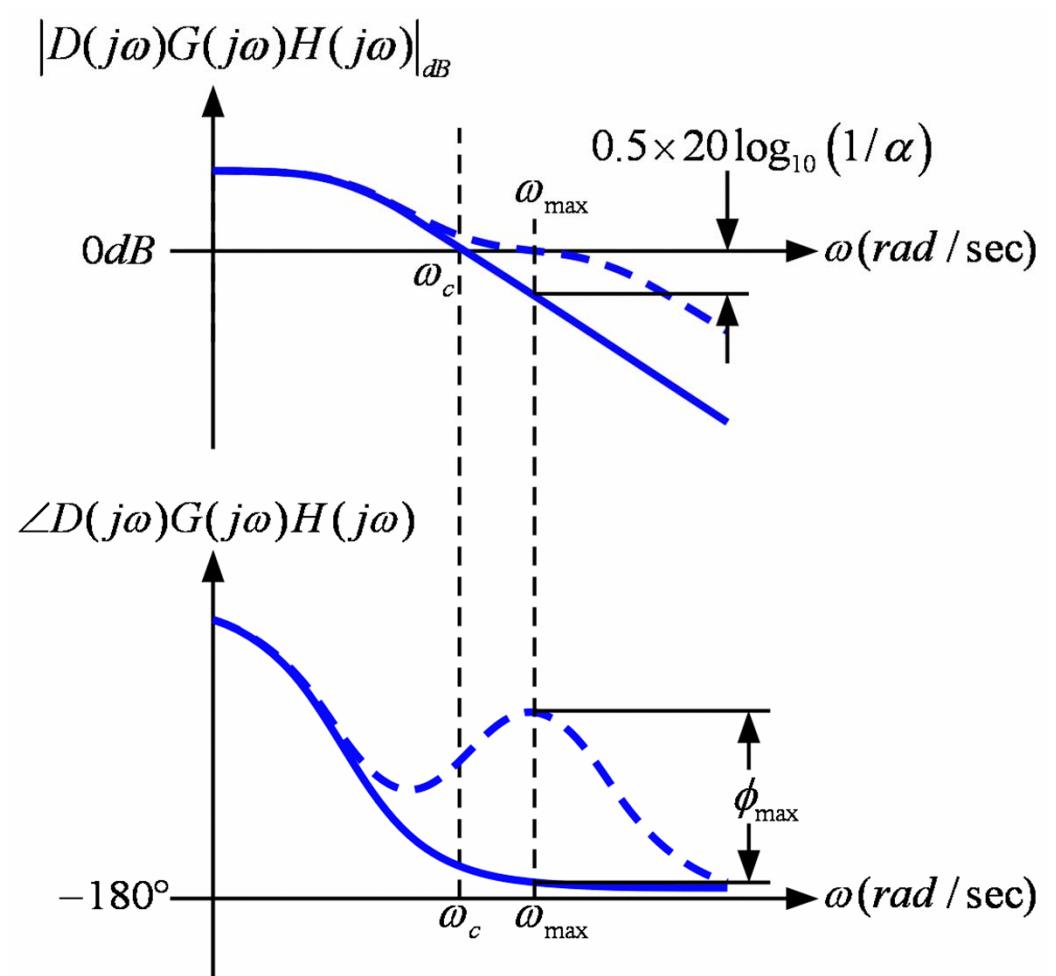
$$\phi_{\max} = \tan^{-1} \frac{1}{\sqrt{\alpha}} - \tan^{-1} \sqrt{\alpha}$$

$$\tan \phi_{\max} = \frac{1-\alpha}{2\sqrt{\alpha}} \quad \sin \phi_{\max} = \frac{1-\alpha}{1+\alpha} \quad \alpha = \frac{1-\sin \phi_{\max}}{1+\sin \phi_{\max}}$$

진상 제어기의 설계

$$20\log_{10}|KG(j\omega_{\max})H(j\omega_{\max})| = -0.5 \times 20\log_{10}\frac{1}{\alpha}$$

$$T = \frac{1}{\omega_{\max} \sqrt{\alpha}}$$



예제 7-2

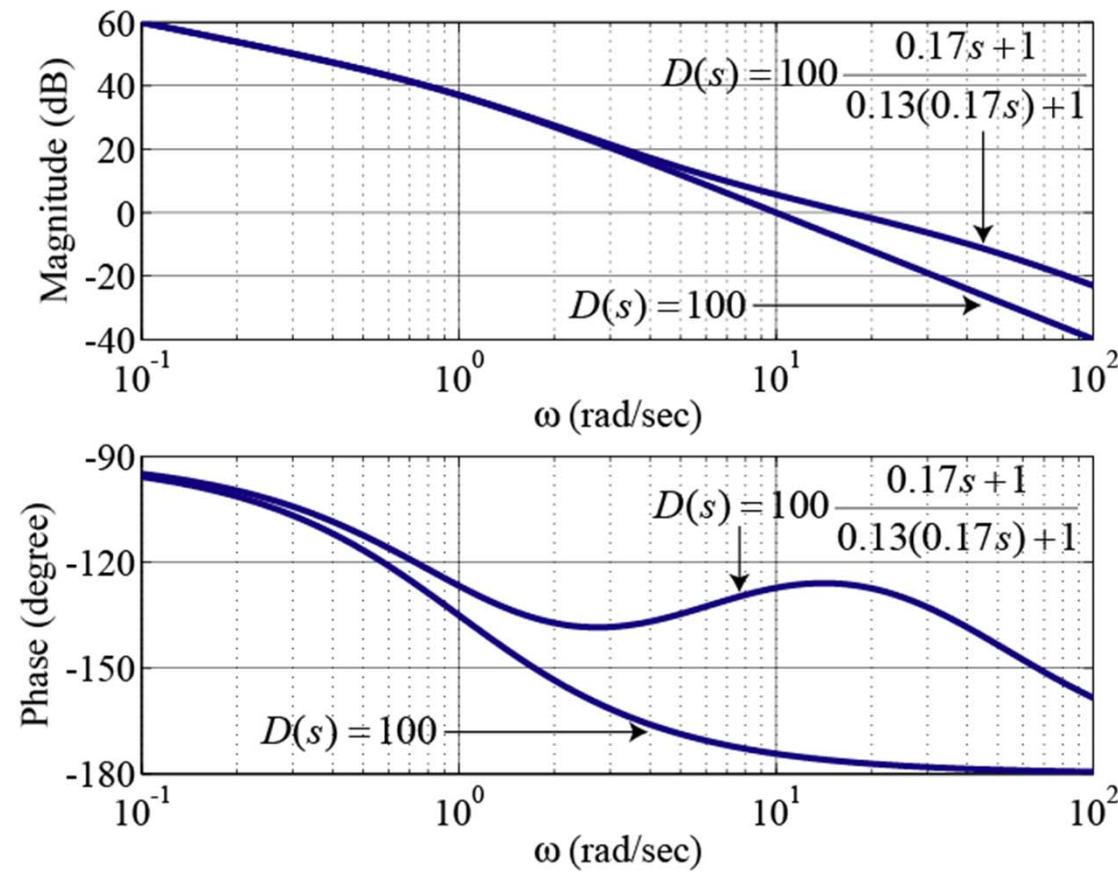
$$\phi_{\max} = 50^\circ \quad \alpha = 0.13$$

$$0.5 \times 20 \log_{10}(1/\alpha) = 9 dB$$

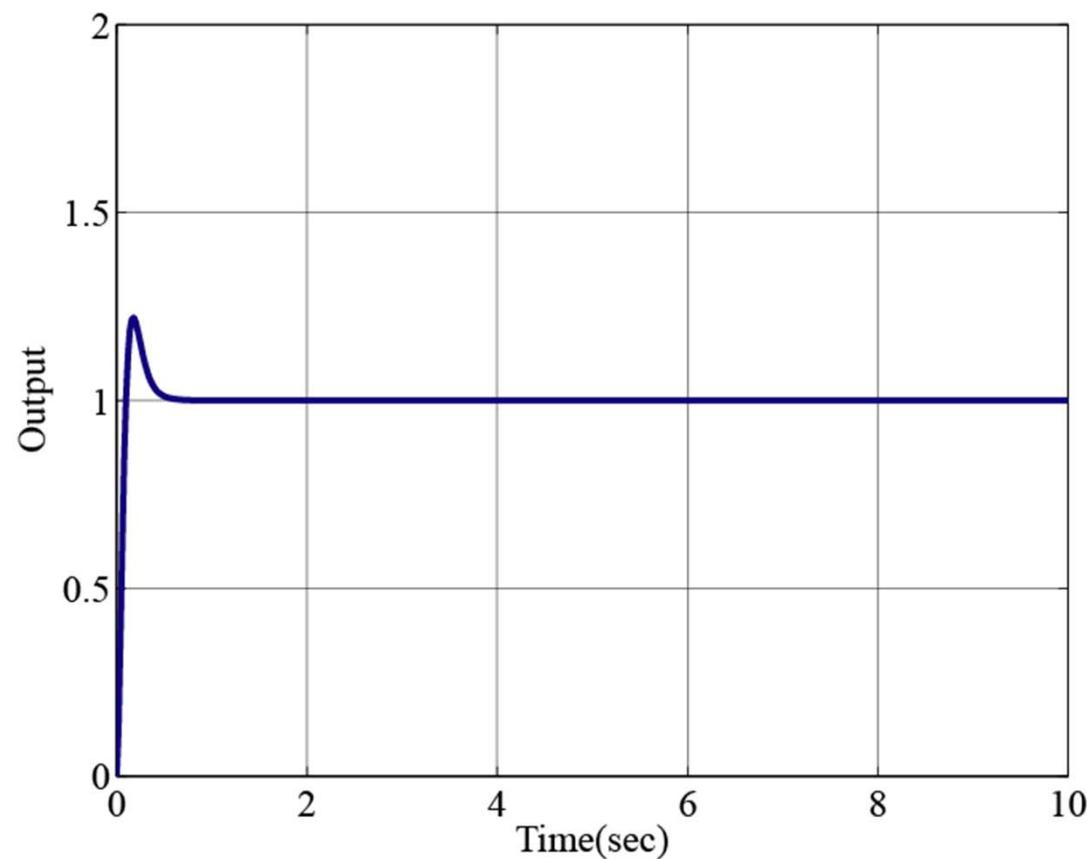
$$T = \frac{1}{\omega_{\max} \sqrt{\alpha}} = \frac{1}{16.7 \sqrt{0.13}} = 0.17$$

$$D(s) = K \frac{Ts+1}{\alpha Ts+1} = 100 \frac{0.17s+1}{0.13(0.17s)+1}$$

예제 7-2



예제 7-2



Convert to Digital Form

$$D(s) = \frac{U(s)}{E(s)} = K \frac{Ts + 1}{\alpha Ts + 1}$$

$$D(z) = \frac{U(z)}{E(z)} = K \frac{Ts + 1}{\alpha Ts + 1} \Big|_{s=\frac{z-1}{T_s z+1}} = K \frac{\frac{T}{T_s} \frac{z-1}{z+1} + 1}{\alpha T \frac{2}{T_s} \frac{z-1}{z+1} + 1}$$

$$u(n) = \frac{1}{T_s + 2\alpha T} \left[K(T_s + 2T)e(n) + K(T_s - 2T)e(n-1) + (2\alpha T - T_s)u(n-1) \right]$$